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SUBJECT: Screening Tests of Mechanical Pipe Joints for a Fused Salt Reactor System

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ABSTRACT

The testing and evaluation of three types of mechanical joints in a circulating molten fluoride salt system, at temperatures up to $1500^{\circ}F_{s}$ has been accomplished.

The feasibility of these joints for use in a large scale molten salt system is discussed.

Design criteria and operating techniques are described.

Measurements have been made of the leakage rates of helium through the joints. The effects of thermal cycling, atmospheric oxidation, salt corrosion, and thermal stresses have been noted. Disassembly and reassembly procedures are described.

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1.0 Introduction

To reduce down-time during maintenance operations on the molten salt power reactor system to a minimum, the philosophy has been adopted that all system components on which direct maintenance is not possible shall be removable and replaceable with spares, by remote manipulation. Repair work on failed components shall be accomplished in suitably equipped independent facilities.

It is prerequisite to the removal and replacement of system components by remote manipulation that a reliable method be developed for parting system piping and rejoining it to its original integrity. Until present experimental efforts to perform critical welding remotely have succeeded to the point where they can be applied with confidence to the maintenance of a reactor system, the use of mechanical joints for this purpose must be considered. The objective of the work reported here was to screen the various concepts of mechanical joints offered by individuals or groups within the Reactor Projects Division, and to select for development those showing promise of successful application.

2.0 Summary

Three small scale mechanical pipe joints of alternative types were tested in a fused salt pump loop. The joints were cold leak checked on a mass spectrometer leak detector before installation in the loop, were then installed and subjected to a series of thermal cycles between 1100 and 1500° F, removed from the loop and leak checked. The joints were then parted, remade, and leak checked again. An indicated leak rate of less than 1 x 10⁻⁷ cc of belium per second was required for acceptable performance at each check.

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Of the three joints tested, the "freeze flange" joint was found satisfactory for immediate development, the "indented seal" joint was found promising if modified, and the "cast seal" joint was found not suitable for development.

In accordance with these results, further tests of the small-scale "freeze flange" and "indented seal" joints have been initiated with sodium as the process fluid, and a pair of "freeze flange" joints in 4" pipe size has been fabricated for testing in a large fused salt system. Results of these tests will be reported separately.

3.0 Description of Test Joints

3.1 Criteria for Joint Performance

For the screening tests, the following criteria were used, based upon the Preliminary Mechanical Joint Specification (see Appendix 1) prepared by the MSRP Group:

(a) The joints shall be fabricated of materials compatible with fused fluoride salt fuels.

(b) The joint leak rate, cold, before installation in the test loop, shall be less than 1×10^{-7} cc of helium per second, as indicated by comparison with a standard leak used in conjunction with a mass spectrometer leak detector.

(c) The joint shall be installed in a test loop circulating fused salt and subjected to a minimum of 50 thermal cycles between the temperatures of 1150 and 1300°F.

(d) No fluid leakage shall be acceptable.

(e) Upon completion of the thermal cycling tests, the joint shall be cut from the loop and cold leak checked (see 3.1.b).

(f) It shall be demonstrated that each joint can be parted and remade acceptably leak tight, without requiring more than superficial cleaning, etc., such as could be performed by remote manipulation.

3.2 Freeze Flange Joint

The principle of the frozen seal mechanical joint, or freeze flange, is illustrated in Fig. 1. The frozen salt seal is formed in the gap between the flange faces. The ring insert provides a labyrinth restriction to the passage of salt into the gap. The labyrinth is important only when the test loop was filled with salt, before the frozen seal was formed. After the frozen seal is formed, it retains the molten salt in the process stream. The metal seal ring provides a gas-tight seal, which retains fission gases. For an effective gas seal, it was found necessary to manufacture the seal ring to close tolerances. The 0. D. and I. D. of the ring were within 0.004 inches on a diameter of seven inches, and concentric within 0.001 inches. The finish required was sixteen microinches. The same tolerances were required in making the seal ring groove in the flange faces.

An air channel, in each half of the flange assembly, provides cooling for the seal ring and insures the formation of the frozen salt seal.

The flange has a thin cross section between the flange hub, which is welded to the loop tubing, and the seal ring, near the outer edge of the flange. This was designed to present a small cross sectional area for radial beat conduction. Four webs add strength to the flange and reduce warping.

On the test installation, eight quick-opening toggle clamps were used on each flange assembly, as shown in Fig. 2, to provide the necessary

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XXXX BETWEEN FLANGE FACES INDICATE REGION OF FROZEN-SALT SEAL; INDICATES REGION OF TRANSITION FROM LIQUID TO SOLID SALT

Fig. 1. Cross Section of Freeze Flange Joint.



Fig. 2. Freeze Flanges Installed in Inconel Test Loop. (1) Air Cooling Lines (2 places); (2) Calrod Heaters (1 place); (3) Toggle Clamp (1 place); (4) Support for Flange (1 place); (5) Clam Shell Heater (1 place); (6) Hy-Temp Insulation (1 place)

force to maintain the gas leakage rate below the maximum allowed.

Fig. 2 shows the assembled flanges with clam shell type heaters, located on the loop tubing immediately before and after each flange assembly. Referring to the figure, the various parts are as follows: (1) Air cooling lines, (2) Calrod heaters leads, (3) Quick-open toggle clamps, (4) Thermocouple leads, (5) Clam shell heaters, (6) Insulation.

There were no heaters nor was there any insulation on the flange assemblies.

The flanges were made of Inconel. The seal rings for the first test were of stainless steel. In subsequent tests, soft iron and annealed copper seal rings were used.

3.3 Indented Seal Joint

This mechanical joint is illustrated in Fig. 3. A dead-soft annealed metal gasket is used to seal the two flange halves. A raised tooth of V-shaped cross section, machined on the face of each flange, indents the flat metal gasket to provide a tight gas seal. Actual parts are shown in Figs. 4 and 5.

A thin cross section between the flange hub and gasket, near the outer edge of the flange, presents a small cross sectional area for radial heat conduction.

On the test installation, four standard design "C" clamps were used on each flange assembly to provide the force necessary to indent the gasket and maintain a gas leakage rate below the maximum allowed. Fig. 6 shows an assembly with clamps.

The flanges were made of Inconel. For the first test, a dead-soft annealed copper gasket was used in one flange assembly, and an annealed

INDENTATION SEAL FLANGE TEST NO. 1. THERMOCOUPLE LOCATIONS



Fig. 3. Schematic Drawing of Indented Seal Mechanical Joint.

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Fig. 4. Assembly of Indented Seal Flange Before Testing.



Fig. 5. Parts of Indented Seal Flange Before Testing.

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Fig. 6. Indented Seal Flange Shown Assembled with "C" Clamps.

Armco iron gasket in the other assembly. For the second test, an annealed Nickel "A" gasket was used in one assembly and a nickel-plated Armco iron gasket in the other assembly.

3.4 Cast Seal Joint

A cross section of this mechanical joint is shown in Fig. 7. Four stainless steel bolts provide the mechanical strength for the joint. A metal insert, shown in the figure before being fused, is used to provide a salt and gas seal. The metal insert is fused between the flange faces before the test loop is filled with salt. The seal whose melting point is above that of the fuel salt, is in the solid state during operation of the test loop.

The sealing surfaces on the flange faces are flash copper plated, and then nickel plated, to promote good wetting by the sealing alloy.¹

Because of its geometry, this joint must be installed in a vertical run of pipe.

The test flanges were made of Inconel. A cast silver seal was used in one flange assembly, while an alloy of 72% silver and 28% copper was used in the other flange assembly.

An assembly of the flange is shown in Fig. 8.

3.5 The Test Loop

The test loop consisted of 25 feet of one-half inch 0. D., 0.045 inch wall thickness, Inconel tubing arranged as shown in Fig. 9. The various parts are identified as follows: (10) Location of flanges under test, (11) Calrod heaters wrapped with stainless steel strip, (12) Clam

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¹Chapter 2.1 MSRP Quarterly Progress Report, Jan. 31, 1958, ORNL 2474 UC-81 Reactors - Power.



Fig. 7. Cast Seat Flange Showing a Cross Section View and Thermocouple Locations.

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Fig. 8. Assembly of Cast Seal Flanges.

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Fig. 9. Inconel Test Loop. (10) Indented Seal Flange (1 place); (11) Calrod Heaters Wrapped with Stainless Steel Strip (2 places); (12) Clam Shell Heaters (1 place); (13) Unistrut Frame (1 place); (14) LFB Pump (1 place); (15) Salt Sump (1 place); (16) Metal Trough Support (1 place); (17) Hy-Temp Insulation (1 place)

shell heaters, (13) Unistrut frame, (14) LFB pump, (15) "Hy-temp" insulation.

The molten salt mixture was circulated by means of an LFB-type centrifugal pump. During tests 1 and 2, on the freeze flanges, two straight sections of the loop, each about nine feet long, were used as resistance heaters. A heavy current was passed through the tubing sections. Other sections of the loop were heated with calrod or clam shell heaters. The entire loop was preheated, prior to filling with salt.

Tests 3 and 4 on the freeze flanges were conducted in a new location where resistance heating was not available, and the entire loop and pump were heated by means of calrod and clam shell heaters.

A sump tank, connected to the point of lowest elevation in the loop, was used to fill and drain the loop of molten salt. The LFB pump, installed at the highest point in the loop also served as a surge tank. The pump was supplied with an oil lubricating system. A helium supply was connected to the loop for initial purging of air before heating. It was also used to pressurize the sump for filling the loop with molten salt. The various components at the pump end of the loop are shown in Fig. 10 and are identified as follows: (1) LFB pump, (2) Pump motor and clutch, (3) Salt sump, (4) Water cooled oil storage tank, (5) Lubricating oil pumps, (6) Helium supply regulators, (7) Helium bubbler, (8) Helium pressure gauges, (9) Lubricating oil flowmeter.

Salt flow in the loop was controlled by a variable speed magnetictype clutch and induction motor. V-belts connected the motor and clutch to the LFB pump.

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- Fig. 10. Inconel Test Loop Pump End. (1) LFB Pump (1 place); (2) Pump Motor and Clutch (1 place); (3) Salt Sump (1 place);
- (4) Water Cooled Oil Storage Tank (1 place); (5) Lube Oil Pumps (1 place); (6) He Supply Regulator (1 place); (7) He Bubbler (1 place); (8) He Pressure Gages (1 place); (9) Lube Oil Flowmeter (1 place).

All components of the loop in contact with the molten salt were fabricated of Inconel.

The loop design was similar to a "standard" design established in the Experimental Engineering Department for their corrosion testing program. The entire loop, including the pump, was mounted on a test stand, also of "standard" design, made of Unistrut. The stand was mounted on wheels for easy transport between various construction stations, such as the weld shop and X-ray room.

All welds in contact with the molten salt were given a dye check and X-ray examination according to departmental specifications for critical welds. The Inconel tubing used in the loop had received a thorough dye check inspection prior to construction of the loop.

Twenty thermocouples were spot welded to the loop tubing and pump, per Dwg. No. SK-CKM-1623. Additional thermocouples were attached to the flanges.

Precautions were taken during construction to use clean tubing and parts, and especially to avoid contamination of the inside of the tubing and pump.

3.6 Instrumentation

The chart below lists instruments and controls used on the test loop for all of the flange tests.

	Measurement	Primary Element	Indicating Element	Control
(a)	Temperature	Chromel-alumel thermocouples	Temperature Re- corders (1)	Tests 1 and 2: (Freeze Flange) Resistance Heaters, Transformer (2), Clam shell heaters, Variacs. Tests 3 and 4: (Freeze Flange and other flange tests) Clam shell heaters.

Variacs (3)

	<u>Measurement</u>	Primary Element	Indicating Element	Control
(Ъ)	Salt levels in pump and sump	Metal probes and 110 V. power supply	110 V. lights	
(c)	Helium pressure for LFB pump and sump	e Bourdo	on Gauges	Pressure regu- lators (4)
(d)	Lubricating oil flow for LFB pump	Flowra	.tor	Centrifugal pump and motor (5)
(e)	Pump speed	Strobo	stac	Motor and clutch (6)

Notes on instruments and controls list:

- (1) Bristol recorders 0-2000°F range for Tests 1 and 2 (Freeze Flange) Brown recorders 0-2000°F range for Tests 3 and 4 (Freeze Flange and other flange tests)
- (2) Resistance heaters consisted of two lengths of 1/2" O. D. x 0.045" wall Inconel tubing each approximately 9 feet long. Hevi-duty transformer and saturable reactor control rated at 110 KVA, which allows a maximum current of 2740 amps output on the 40 volt tap; 3 KW is the minimum leakage power; 20 KW output can be controlled, but it is difficult to provide steady control of a lower power. Wheelco controller

Vickers magnetic amplifier

- (3) Open element type cylindrical clam shell heaters. Variacs, 110 V. and 220 V.
- (4) Fisher Governor bleed and non-bleed types of pressure regulators, 0-35 psig, Models 67-15 and 67-16.

- (5) Eastern Industries Model D-11 centrifugal pump with 1/5 H. P.,110 V. motor.
- (6) Louis Allis Adjusto-Spede induction motor, 5 H. P., 3 phase, ~3600 rpm.
 Dynamatic magnetic-type clutch, 400-3400 rpm at full load.

4.0 Test Procedure

4.1 Freeze Flange Joint

Two sets of flanges were assembled with seal rings prior to installation in the loop. Each flange assembly was then leak tested using a helium leak detector. To leak test, the tubing from one end of a flange assembly was plugged and the other end connected to a vacuum pump. A plastic bag was then placed around the flange assembly and kept filled with helium during the leak test. Helium leak rate tests were made on both flange assemblies prior to installation in the loop for Tests 3 and 4. This work was done by the Instrument Department and is described in detail in their report No. 58-1-20. Both flange assemblies were then welded into one end of the loop as shown in Figs. 2 and 9.

Helium was circulated through the loop before preheating, to remove air. A helium pressure was maintained on the loop and pump during heating and filling.

Clean fluoride salt mixture was then transferred to the sump tank. When the temperature of all parts of the loop reached 1300°F, helium pressure was applied to the sump tank to force the salt into the loop. The pump was operated at a low speed during the filling operation. The loop was vented through the pump helium inlet, as the salt displaced the helium. Salt circulation was obtained a few minutes later, when the pump speed was increased to 2500 rpm to produce about 2 gpm flow. A helium blanket was maintained at 2 to 3 psig on the salt surface in the pump during testing operations.

The heaters were then adjusted to obtain 1300°F on all parts of the loop. During the loop filling operation, cooling air was circulated in the air channels built into the freeze flanges. A small air flow of approximately 15 cfh was necessary to keep the seal ring area cool.

The loop was filled rapidly to insure establishment of frozen seals between the flange faces before the molten salt could raise the temperature of the flanges.

After a period of isothermal operation to make certain there was no salt leak, the thermal cycle testing was begun. Temperatures of the salt were measured by means of thermocouples located on the loop tubing at the inlet to each flange assembly. Temperature measurements were taken at various points on each flange assembly.

The salt flow was maintained at a steady rate of 2 gpm. The pump speed was checked periodically with a Strobotac and the flow in gpm obtained from a pump calibration chart.

A schematic diagram of the Inconel loop during Tests 1 and 2 is shown in Fig. 11. During these tests, temperatures were measured at the points indicated in Fig. 12. Fluoride salt, Fuel No. 107, was circulated in the loop during the first test. The salt temperature was cycled 50 times between 11.50°F and 1300°F, and then the loop was operated isothermally for several days at 1300°F. The average cycle time was 44 minutes. A cycle is defined as a temperature variation from 1300 to 1150°F and back to 1300°F.

During the second test, in which fluoride salt, Fuel No. 30, was used, the temperature variation of the salt was from 1300 to 1500°F and back to



Fig. 11. Schematic Drawing of Test Loop for Freeze Flange Tests Land 2.



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Fig. 12. Schematic Drawing of Thermocouple Locations on the Freeze Flanges for Tests 1 and 2.

1300°F. The number of cycles was reduced to 30 and the average cycle time was 29 minutes.

After Tests 1 and 2, the flange assemblies were disassembled and reassembled to demonstrate ease and speed of handling. A motion picture film was taken of these operations. The time required for these operations is listed in Table 1.

Both of the flange assemblies were removed from the loop before starting on the next test. Additional machining work was performed on the flange faces to produce a better finish on the seal ring grooves.

A schematic diagram of the Inconel loop for Tests 3 and 4 is shown in Fig. 13. Fluoride salt, Fuel No. 30, was used in these tests. The temperatures were measured at the points indicated in Fig. 14. The salt flow rate was, again, 2 gpm.

During Test No. 3 the salt temperature was cycled 50 times between 1100°F and 1300°F. The average cycle time was 60 minutes. During Test No. 4 the temperature variation of the salt was from 1100 to 1300°F and back to 1100°F; the number of cycles was 25, and the average cycle time was 62 minutes.

After the thermal cycling tests, both flange assemblies were removed from the loop for leak testing. Both assemblies were disassembled and one flange assembly was reassembled using a new copper seal ring. This assembly was again leak tested.

4.2 Cast Seal Joint

Two sets of flanges were assembled with metal inserts prior to installation in the loop. One flange assembly contained a pure silver insert, the other assembly contained an insert of copper-silver alloy. Both

Table	No.	1
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	TIME REQUIRED FOR ASSEMBLY ANI	DISASSEMBLY	FOR FREEZE	FLANGES	3
	DISAS	SEMBLY	and the second	ацан. 	A
	Operation Performed			Time	Required
ı.	Dump Salt from Loop			15	min.
2.	Cooling Flanges to Room Temperature			4	hrs.
3.	Disconnect Thermocouples - Heaters - Airlines			1.5	min.
4.	Remove Clamps			3.5	min.
(Se	e Figs. 21 and 22 for view of disassembled flan	ges)	Total	4	hrs. 20 min.
	ASSE	MBLY			
l.	Cleaning Flanges			5 :	min.
2.	Replacing Flanges and Clamps			5.5 :	min.
3.	Reconnection of Thermocouples - Heaters - Airl	ines		3 1	min.
(50			Total	13.5	min.

(See Fig. 2 for view of assembled flanges)

Note:

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The time periods listed are for handling two freeze flange assemblies with two men performing the operations. The time periods were taken from a motion picture of the operations.



Fig. 13. Schematic Drawing of Test Loop for Freeze Flange Tests 3 and 4 and Cast Seal Flange Tests 1 and 2 .



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Fig. 14. Schematic Drawing of Thermocouple Locations on the Freeze Flanges for Tests 3 and 4.

flange assemblies were then welded into one end of the loop in the location shown in Fig. 13.

A special furnace was placed around each cast seal flange for melting the metal inserts and to provide heat during loop operation. The loop and flanges were filled with helium prior to heating. The assembly with the silver insert was raised to 1830 - 1975°F and held at this temperature for one hour to insure complete fusion of the silver. The melting point of silver is 1760°F. The other assembly was treated in the same fashion, except that the temperature was raised to 1570° - 1670°F. The melting point of the copper-silver alloy is 1435°F.

The loop and flanges were then filled with helium at a pressure of approximately 25 psig to check for leakage through the cast metal seals. The pressure did not drop after several hours, and the flanges gave no indication of any leakage when checked with a scap solution coated over the flange joint and around the bolts.

The loop was then placed in operation for thermal cycle testing as described in Section 4.1.

During these tests, temperatures were measured at the points indicated in Fig. 7. Fluoride salt, Fuel No. 30, was circulated in the loop during both tests on the cast seal flange. During Test No. 1, the salt temperature was cycled 50 times between 1300 and 1100°F. The average cycle time was 60 minutes. During Test No. 2, the salt temperature variation was from 1100°F to 1300°F and back to 1100°F. The number of cycles was 25 and the average cycle time was 62 minutes. Tests 1 and 2 on the cast seal flanges were run simultaneously with Tests 3 and 4 on the freeze flanges.

After the cycling tests, both flange assemblies were separated using the special furnaces to melt the cast seals during the parting operation. Stainless steel rods were tack welded to the flanges to pull them apart, while the seals were in the molten state. A diagram of this apparatus is shown in Fig. 15.

The top halves of each of the two flange assemblies were thoroughly cleaned by placing them in a hydrogen furnace at 1900°F for about two hours. The bottom halves of both assemblies were given a superficial cleaning using a brush and a vacuum cleaner. The bottom halves of the flanges represent those parts of the mechanical joints which could not be removed from a reactor system for cleaning and would have to be cleaned remotely.

Both flange assemblies were then reassembled using spring-loaded tie rods to apply the necessary force to push the flange halves together. A diagram of this apparatus is shown in Fig. 16. The special furnaces were used around each assembly to melt the cast seals during the reassembly operation.

Both flange assemblies were removed from the loop and sectioned for metallurgical examination. This work was done by the Metallurgy Division. (See Appendix 5 for their report.)

4.3 Indented Seal Joint

Two sets of flanges were assembled with metal gaskets prior to installation in the loop. Four "C" clamps were used on each flange assembly. The clamps were each tightened with a torque of 80 - 90 foot pounds. The load on each gasket was approximately 12,000 pounds per inch of gasket circumference. Each flange assembly was leak tested using a helium leak detector. Both assemblies were then welded into one end of the loop as shown in Figs. 9 and 17.

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Fig. 15. Sketch of Extension Rods for Disassembly of Cast Seal Flanges.



Fig. 16. Sketch of Spring-Loaded Tie Rods for Reassembly of Cast Seal Flanges.



Fig. 17. Schematic Drawing of Test Loop for Indented Seal Flange Tests 1 and 2.

The loop was then placed in operation for thermal cycle testing as described in Section 4.1. During Test No. 1 temperatures were measured at the points indicated in Fig. 3. Fluoride salt, Fuel No. 30, was circulated in the loop at a flow rate of 2 gpm. The salt temperature was cycled 50 times between 1100°F and 1300°F. The average cycle time was 60 minutes.

Both flange assemblies were removed from the loop after Test No. 1 for leak testing. Subsequently, the assemblies were separated and reassembled with new gaskets. The flanges were then re-installed in the loop for Test No. 2.

During this test, temperatures were measured at the points indicated in Fig. 18. The salt temperature was cycled between 1100 and 1300°F. The number of cycles was 50 and the average cycle time was 61 minutes.

Again, the flanges were removed from the loop for leak testing and disassembly. One of the flange assemblies was reassembled with a new gasket and then leak tested.

5.0 Discussion of Results

5.1 Freeze Flange Joint

The maximum and minimum temperatures of various locations on the flanges during thermal cycling are listed for Tests 1 and 2 in Table 2; for Tests 3 and 4 in Table 3.

Examination of Tables 2 and 3 will show that the greatest temperature variation, at any one point in the flanges, occurs at the flange hub near the loop tubing. For example, thermocouple No. 21, Table No. 2, Test No. 1, varied from 862 to 1018°F. Thus, the greatest temperature variation, at this point during cycling, was 156°F. The smallest variation occurs at the periphery of the flanges as indicated, in one case, by thermocouple No. 24 in Table No. 2. The minimum and maximum temperatures indicated

INDENTATION SEAL FLANGE TEST NO. 2. THERMOCOUPLE LOCATIONS



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FLANGE A-NICKEL PLATED IRON GASKET TC NO. 27, 28, 29, 30, 31, 32 FLANGE B-NICKEL GASKET TC NO. 33, 34, 35, 36, 37, 38

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Fig. 18. Schematic Drawing of Thermocouple Locations for Test 2 on Indented Seal Flange.

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		Test No 50 Ti	. 1 - Salt mes Between	Temperature Cyc. 1150 and 1300°	led F	Test N 30 T	c. 2 - Salt imes Between	Temperature Cy 1300 and 1500	vcled)°F	
Thermoco Numbe	uple r *	Minimum Temperature (°F)	Cycle Number	Maximum Temperature (°F)	Cycle Number	Minimum Temperature (°F)	Cycle Number	Maximum Temperature (°F)	Cycle Number	
21		862	16	1018	33 , 50	960	3	1170	21,24	_
22	Flange	625	16	760	33,50	695	1,3	950	9	
23	A	450	6,14,20	577	16	500	3	652	9	
24		245	16	280	50	275	3	305	30	
25		693	10	820	4	810	6,12	942	1	
26	Flange	490	16	580	<u>}</u>	582	6,21	665	l	1 \\
27	A	348	16	403	50	410	6	452	1,9,27,30	ۍ ۱
28		205	16	240	4	250	1,3,6	260	9,24	
13		798	6	957	33	915	3,6	1122	21	
14 14	Flange	560	16	685	41	660	1,3	905	27	
15	В	375	16	447	41	465	6	595	15,21,27	
16		200	10,16	228	37	255	24	305	27	
17		748	16	865	2	865	12,18,21	1010	1,3	
18	Flange	510	16	590	2	615	12,18	700	1,6	
19	В	315	16	362	50	410	6	450	3,6,9,30	
20		185	16	215	50	260	21	278	30	

FREEZE FLANGE TEMPERATURES MEASURED DURING THERMAL CYCLES

* (See Fig. 12 for location of thermocouples) (See Fig. 11 for diagram of Inconel loop)

Test No. 1 - Stainless Steel Seal Rings Test No. 2 - Soft Iron Seal Rings

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FREEZE FLANGE TEMPERATURES MEASURED DURING THERMAL CYCLES

T 5	Test No. 3 Salt Temperature Cycled 50 Times Between 1100°F and 1300°F					Test No. 25 Times	Test No. 4 Salt Temperature Cycled 25 Times Between 1100°F and 1300°F			
Thermo No.	couple *	Minimum Temp.°F	Cycle No.	Maximum Temp.°F	Cycle No.	Minimum Temp.°F	Cycle No.	Maximum Temp.°F	Cycle No.	
27	Flange	185	14	218	49	195	1	225	24	
28	Β	340	18	402	1-41-42-49	347	12	400	2-4-24	
29		815	6-7-11	973	l	788	12	957	2	
30	Flange	785	6	947	46	768	12	932	1-2	
31	в	277	15	340	46-48-50	285	12	335	24-25	
32		182	14-15	225	50	190	1-12	218	24-25	
33	Flange	183	15	220	47-50	190	ı	218	22-23-24-25	
3 ⁴	A	295	15	355	50	300	l	352	24	
35		788	6	975	15	760	21	945	2	
36	Flange	737	13	903	46	712	12	885	l	
37	A	287	15	350	46 thru 50	292	12	345	25	
38		190	13-14-15	235	46	200	1-12	233	24-25	

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* (See Fig. 14 for location of thermocouples) (See Fig. 13 for diagram of Inconel loop)

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Tests Nos. 3 and 4 - Copper Seal Rings

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by this thermocouple were 245 and 280°F, respectively. Thus, the greatest temperature variation, at this point during thermal cycling, was 35°F.

The low maximum temperature, and the slight variations in temperature encountered in the region of the fastenings and seal ring during thermal cycling, are desirable features of the freeze flange joint.

The maximum radial temperature difference between the outside and inside diameters of the flange, during the same test, was 738°F. This figure is the difference between thermocouples No. 21 and No. 24, taken at their maximum values. The corresponding salt temperature was 1300°F. The maximum such temperature difference obtained for any of the tests was 817°F.

The above examples are typical of the freeze flange operation. Complete data for all tests is presented in graphical form. The points of large and small temperature variation can be seen to follow the above description. Graphs showing temperatures of the flanges throughout the periods of thermal cycling are included in Appendix 2.

Gas leakage rates for Tests No. 1 and 2 were not satisfactory. This was attributed to defects introduced during the initial fabrication of the flanges. Some welding was performed on the flanges after all of the machining had been completed, which caused a slight warping. In addition, the finish on the seal ring grooves was not fine enough. These defects were remedied by additional machining. Subsequent leakage rates were satisfactory.

Gas leakage rates of the joints used in Tests 3 and 4 are listed in Table 4. One leak rate was obtained on each flange before Test 3 and after Test 4. As can be seen from the data, there was an appreciable increase in

- 37 -

		LEAK	RATES OF FREEZE	E FLANGES ANI) INDEMTED SE	AL FLANGES	
	Free	eze Flanges			Indented	Seal Flanges	
Test No.	. Flange No	Leak Rate D. Before Test	in cc/sec After Test	Test No.	Flange No.	Leak Rate Before Test	in cc/sec After Test
3	А	3×10^{-9}	Not Removed	1	А	3.2 x 10 ⁻⁸	2.2 x 10 ⁻⁸
3	В	2.3 x 10 ⁻⁸	Not Removed	1	В	1 x 10 ⁻⁹	1.5 × 10 ⁻⁹
4	А	Same as Test 3	$1.7 \times 10^{-7} *$	2	А	1.2 x 10 ⁻⁸	3.9 x 10 ⁻⁹ ***
4	В	Same as Test 3	5×10^{-6}	2	В	1.9 x 10 ⁻⁸	1.2 x 10 ⁻⁸ **
(See Rep Leak De	port ORNL CF stection" by	No. 58-1-20, "No [.] H. J. Metz)	tes on Helium	Test No	. l: Flange Flange	A - Iron Gask B - Copper Ga	et sket
(* This	flance secon	har was severated	3 om 3 3 3	1 m	-		

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(* This flange assembly was separated and reassembled using a new copper seal ring to give a leakage rate of 2.0 x 10^{-8} cc/sec)

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- Test No. 2: Flange A Nickel-Plated Iron Gasket Flange B - Nickel Gasket
- (** This result was obtained after tightening one of the "C" clamps. The clamp had loosened when the loop was cooled to room temperature following cycling tests.)

(*** This flange assembly was separated and reassembled with a new nickel-plated iron gasket to give a leakage rate of 5.0 x 10^{-8} cc/sec)

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the leak rate after testing. A third leak rate was obtained on one of the flange assemblies following the cycling tests, after disassembly and reassembly with a new copper seal ring. This leak rate was 2.0×10^{-8} cc/second and successfully met the specification listed in Section 3.1, paragraph (f).

The flange assemblies were heated or cooled between room temperature and the operating temperature of 1300°F at least eight times in the preparation for thermal cycling tests. The rate of cooling and heating was at least 250°F per hour.

Both flanges operated successfully in that there was no indication of salt leakage, the gas leakage rates were satisfactory, and there was no indication of any cracks due to thermal stresses. (See Section 3.1). A memorandum concerning the metallurgical examination for stress cracks is included in Appendix 6.

Oxidation of the copper seal rings was negligible. The rings had the same bright appearance after testing as they did originally. The Inconel flanges showed no noticeable salt corrosion. The flanges and seal rings are shown after the completion of Test No. 4 in Figs. 19 and 20.

The formation of the salt seal is shown in Fig. 21 after thermal cycling tests and disassembly. Parts are identified as follows: (1) Frozen salt seal, (2) Labyrinth insert ring, (3) Groove for seal ring. Both flanges are shown after disassembly in Fig. 22 and various parts identified as follows: (1) Cooling air lines, (2) Resistance heater lugs, (3) Calrod heater leads, (4) Thermocouple leads, (5) Clam shell heaters, (6) Copper seal rings in place.

The same flange is shown after removal of the salt seal in Fig. 23. The time for cleaning both flanges was not more than 5 minutes.



Fig. 19. Freeze Flange (1/2" Tubing Size) Open with Copper Seal Ring After Cleaning, Following Test 4.



Fig. 20. Copper Seal Ring from Small Freeze Flange After Test 4.



Fig. 21. Freeze Flange Showing Formation of Frozen Salt Seal.



Fig. 22. Freeze Flanges Installed in Test Loop After Disassembly of the Flanges. Photo showing both flanges disassembled.

(1) Air Cooling Inlet (4 places); (2) Resistance Heater Leads (2 places); (3) Calrod Heater

- (1 place); (4) Thermocouple Leads (2 places); (5) Clam Shell Heaters (2 places)
- (6) Seal Ring (2 places)



Fig. 23. Freeze Flange After Removal of Salt Seal from Face of Flange.

The time necessary for the handling operations of disassembly and reassembly is indicated in Table 1. Ease and speed of handling were demonstrated, indicating that the freeze flange joint is potentially suited to remote manipulation. Other tests are in progress to demonstrate remote handling of this type of joint.

5.2 Cast Seal Joint

The maximum and minimum temperature of various locations on the flanges during thermal cycling are listed, for Tests 1 and 2, in Table 5.

Examination of Table 5 shows that the temperature variation, during thermal cycling, is nearly the same for a thermocouple located in a well near the cast seal as it is for a thermocouple located at the flange periphery near the flange bolts. For example, thermocouple No. 25, near a bolt, varied from 1177°F to 1287°F during Test No. 1. This is a variation during thermal cycling of 110°F. During the same test thermocouple No. 23, in a well near the cast seal, varied from 1157°F to 1296°F, a variation of 139°F.

The temperature variation, during thermal cycling, in the region of the bolts is about three times that obtained in the case of the freeze flanges (See Section 5.1). In addition, the bolts were about 1000°F higher in temperature than the clamps on the freeze flanges.

The radial temperature difference, however, between the outside and inside diameter of the flanges is small compared to the freeze flanges, causing less thermal stress.

The above examples are typical of the cast seal flange operation. Complete data for all tests are included in graphical form. Graphs showing temperature of the flanges throughout the periods of thermal cycling are included in Appendix 3. There was no indication of any salt leakage during

CAST	SEAL	FLANGE	TEMPERATURES	MEASURED	DURING	THERMAL	CYCLES

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Test No. 1 Salt Temperature Cycled 50 Times Between 1100°F and 1300°F					Test No. 2 Salt Temperature Cycled 25 Times Between 1100°F and 1300°F			
Thermocouple No. *	Minimum Temp.°F	Cycle No.	Maximum Temp.°F	Cycle No.	Minimum Temp.°F	Cycle No.	Maximum Temp.°F	Cycle No.
19	1158	13	1290	1-41	1128	20	1255	24
20 Note	1150	13	1280	1-41-42	1122	20	1245	15-24
21 (1)	1212	13	1303	41	1155	20	1238	15
22	1188	6-13	1323	42	1142	20	1285	24
23	1157	6-13	1296	41	1125	20	1258	15-24
24 Note	1142	13	1282	41	1110	20	1243	15-24
25 ⁽²⁾	1177	1 -13	1287	41	1130	20	1233	15-24
26	1137	6	1 31 5	42	1110	20	1293	24

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Notes:

- (1) Flange with Silver Cast Seal
- (2) Flange with Copper-Silver Alloy Cast Seal
- (3) The salt and flange temperatures before cycling were 1300°F
- * (See Fig. 7 for thermocouple location) (See Fig. 13 for diagram of Inconel loop)

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Note:

The actual minimum temperature of all thermocouples in Test No. 2 was during the lst cycle and was from 10° to 45° lower than the minimum temperatures listed in table above.

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The salt and flange temperatures before cycling were 1100 °F.

any of the testing. There was no indication of gas leakage with the loop and flanges under 25 psig of helium, following the initial fusion of the ring inserts. The pressure indication on the loop did not decrease over a period of several hours. There was no indication of gas leakage using a soap solution on the flanges.

The steps and time periods involved in disassembly of these flanges are listed in Table 6. Difficulty was encountered in step (5), the removal of four stainless steel bolts and nuts from each flange. Each nut had to be heated to a red heat, and a wrench used with considerable force. In step (9), separation of the flanges while hot, it was found that the flanges could not be pulled apart with a 25 to 50 pound force on the extension rods connected to the upper flange half. It was necessary to open the heater box around each flange and use a cold chisel to separate the flange halves. Since all of the disassembly operations would be accomplished remotely in a reactor system, it is clear that the cast seal joints, as tested, were not satisfactory in this respect. A comparison may be made in the time necessary for disassembly of the Freeze Flange Joint and Cast Seal Joint by referring to the data in Tables 1 and 6.

The steps involved in reassembly of both flanges are shown in Table 6. Steps 1, 2, 3 and 6 would be accomplished remotely in an actual reactor installation. After reassembly, it was found that both assemblies leaked helium badly at the interfaces of the flange halves. Leaks were detected with a scap solution while the flanges were under 15 psig of helium pressure.

Photographs taken after disassembly of the flanges show that the flanges and sealing alloys were badly oxidized. Fig. 24 shows the flange with the copper-silver alloy seal; Fig. 25 the flange with the silver seal.

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Fig. 24, Cast Seal Flange After Disassembly Following Loop Tests – Copper–Silver Alloy Seal.



Fig. 25. Cast Seal Flange After Disassembly Following Loop Tests - Silver Seal.

TIME REQUIRED FOR ASSEMBLY AND DISASSEMBLY FOR CAST SEAL FLANGES DISASSEMBLY

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				Time Req	uired	
	Operation Performed	<u>s</u>	<u>il</u>	ver Seal	Copper Silv	er Alloy Seal
l.	Dumping Salt from Loop		15	min.	15	min.
2.	Cooling Flange to Room Temperature		4	hours	4	hours
3.	Removal of Insulation from Heater Box		30	min.	30	min.
4.	Removal of Heater Box from Flanges		10	min.	10	min.
5.	Removal of Four Bolts and Nuts from Flanges		40	min.	20	min.
6.	Reinstall Heater Box		10	min.	10	min.
7.	Replace Insulation		20	min.	20	min.
8.	Heat Flanges to Melt Point of Seals	6	•5	hours	4.5	hours
9.	Separation of Flanges while Hot		20	min.	10	min.
10.	Cooling Flanges to Room Temperature		3	hours	2	hours
		Total	16	hours	12	hours
		ASS	EM	BLY		
l.	Cleaning Oxide from Flanges		10	min.	10	min.
2.	Assemble and Install Flanges in Position		30	min,	30	min.
3∘	Install Heater Box and Insulation		15	min,	15	min.
4.	Heat Seals to Melt Point	24	• 5	hours	4.5	hours
5۰	Heat Flanges to Compress and Adjust	3	•5	hours	3	hours
6.	Install Four Bolts and Nuts		5	min.	5	min.
		Total	9	hours	8.5	hours

(See Fig. 8 for view of assembled flange)

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Metallurgical examination confirmed that damage was done to the metal seals during the heating operations. See Appendix 3.

5.3 Indented Seal Joint

The maximum and minimum temperatures of various locations on the flanges during thermal cycling are listed, for Test No. 1 in Table 7; for Test No. 2 in Table 8.

Examination of Table 8 shows that the temperature variation, during thermal cycling, for a thermoccuple located in a well near the metal gasket is about one-half that of a thermoccuple located at the hub of the flange near the loop tubing.

For example, thermocouple No. 30, near the gasket, varied from 960°F to 1065°F, a variation during cycling of 105°F. Thermocouple No. 27 varied from 1090°F to 1310°F, a variation during cycling of 220°F.

The temperature variation, during thermal cycling in the region of the clamps, is about three times that obtained in the case of the freeze flange (See Section 5.1). This factor, together with the high temperature in the region of the fastenings, means that the clamps on the Indented Seal Flange would have a greater tendency to loosen than those on the freeze flange.

The radial temperature difference, however, between the cutside and inside diameter of the flanges is about 1/3 that of the freeze flanges causing less thermal stress.

The above examples are typical of the indented seal flange operation. Complete data for all tests are presented in graphical form. Graphs showing temperatures of the flanges throughout the periods of thermal cycling are included in Appendix 4.

Gas leakage rates are listed in Table 4. Leak rates were obtained

INDENTATION SEAL FLANGE TEMPERATURES MEASURED DURING TEERMAL CYCLES - TEST NO. 1

Thermocoupl Number	. e	Micimum Temperature (°F)	Cycle Number	Maximum Nemperature (°P)	Cycle Number
27		1073	10-	1320] 7
28		778	10	930	, ,
29	Flange	620	10	713	1
30	А	632	10	735	1
31		733	10	887	1 - 2
32		1040	10	1282	17
33		1082	10 - 36	1317	17
34		847	36	1015	÷1 6
35	Flance	695	36 - 45	803	17
36	B	673	45	780	17
37			*	ૠ	- , *
38		1070	36	1290	17

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Test No. 1 - Salt Temperature Cycled 50 Times Between 1100° and 1300°F

Flange A - Iron Gasket

Flange B - Copper Gasket

* Thermocouple damaged - count not repair during operation

(See Fig. 3 for thermocouple location) (See Fig. 17 for diagram of Inconel loop)

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Thermocouple Number		Minimum Temperature (°F)	Cycle Number	Maximum Temperature	Cycle
27 28 29 30 31 32	Flange A	1090 935 971 960 957 1077	31 28 - 44 12 48 44 31	(*F) 1310 1040 1075 1065 1050 1295	<u>Number</u> 45 13 39 39 39 - 45 45
33 34 35 36 37 38 lange A - Nicl	Flange B kel Plated	1097 93 8 912 927 958 1076 Iron Gasket	44 31 - 44 28 - 44 42 - 44 28 31	1310 1031 1008 1025 1065 1295	37 45 39 39 45 45

INDENTATION SEAL FLANGE TEMPERATURES MEASURED DURING THERMAL CYCLES - TEST NO. 2 Test No. 2 - Salt Temperature Cycled 50 Times Between 1100"

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(See Fig. 18 for thermocouple location) (See Fig. 17 for diagram of Inconel loop)

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Table No. 8

before and after each thermal cycling test. A new gasket was then installed in each flange assembly, followed by another leak test. None of the four gaskets, each of a different material, leaked salt. The flange with the copper gasket leaked helium following the thermal cycling in Test No. 1; however, the leak sealed itself before the flange assembly was removed from the loop for checking with a helium leak detector. The copper gasket was badly oxidized as well, as shown in Fig. 26.

The flange with the annealed Armco iron gasket passed gas leakage tests satisfactorily. This flange and gasket is shown after test in Fig. 27. The iron gasket, after removal from the flange, is shown in Fig. 28. The outer edge of the gasket was oxidized and therefore considered unsuitable.

As can be seen from photographs Nos. 26 and 27, a ring of frozen salt had formed between the flange faces. This frozen seal extended from the loop tubing diameter of 0.410 inches to an outside diameter of approximately 1-1/4 inches. For subsequent tests a heater was placed around the outside of each flange assembly to insure that the salt between the flange faces would be in the molten state during testing.

The flange with the annealed nickel "A" gasket leaked gas after thermal cycling testing; however, this was caused by loosening of one of the "C" clamps, which held the flange halves together. This flange and gasket is shown after testing in Fig. 29.

The flange with the nickel plated Armco iron gasket passed gas leakage tests satisfactorily. It is shown after testing in Fig. 30. No exidization of the gasket was apparent.

With the exception of the annealed Armco iron gasket, removal of the gaskets from the flanges was difficult. It was necessary to cut the gaskets

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Fig. 26. Indented Seal Flange with Copper Gasket After Testing.



Fig. 27. Indented Seal Flange with Armco Iron Gasket After Testing.

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Fig. 28. Armco Iron Gasket After Testing in Indented Seal Flange.



Fig. 29. Indented Seal Flange with Nickel Gasket After Testing.

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Fig. 30. Indented Seal Flange with Nickel-Plated Armco Iron Gasket After Testing.

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free of the flanges in a lathe.

Of the four gaskets tested, the nickel plated and annealed Armco iron gasket proved most satisfactory. The Inconel flanges showed no sign of damage from corrosion or mechanical defects. A more suitable method of clamping is in order, to insure a tight seal during the various temperature changes.

6.0 Future Tests

The same freeze flange joint and indented seal joint, described in Section 3 of this report, are now under test with molten sodium in an Inconel loop. An annealed copper seal ring is being used in the freeze flange. A nickel plated, annealed Armco iron gasket is being used in indented seal flange. The tests are similar to those described above. Both flange assemblies are shown installed at one end of the Inconel loop in Fig. 31.

A larger size freeze flange joint has been fabricated. It is shown assembled in Fig. 32. The flange is shown open with the copper seal ring in place in Fig. 33. The principle of operation is identical to the small size flange tested and described above. Provision has been made on the ring seal for monitoring the gas leak rate during operation. Two such flanges have been fabricated for testing with molten salt in a loop, shown schematically in Fig. 34. The welding necks on the flanges will match a 4-inch, schedule 40 pipe. An unusual feature of the large flange is the use of a special clamp, made in two sections, which holds the flanges around their entire circumference. The clamp is shown in the assembly photograph, Fig. 32. Holes were placed in the webs to reduce heat conduction from the salt to the seal ring area. A metal screen will be inserted in the space between the flange faces to reduce the hold-up volume of salt,

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Fig. 31. Freeze Flange and Indented Seal Flange Installed in Test Loop for Cycling Tests with Molten Sodium. (1) Indented Seal Flange; (2) Calrod Heaters; (3) Toggle Clamp; (4) Air Cooling Inlet; (5) Freeze Flange

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Fig. 32. Assembly of Large Freeze Flange.

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Fig. 33. Large Freeze Flange Shown Open with Copper Seal Ring in Place.

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Fig. 34. Schematic Drawing of Test Loop for Large Freeze Flange.

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and to hold the salt cake together for easy and quick removal when the flanges are disassembled.

This 4-inch joint is made of Inconel, and the seal ring of dead-soft annealed copper with a one mil nickel plate. The clamps are of carbon steel. Two 7/8-inch heat treated bolts are used to hold the clamps and flange halves together.

It is believed that test results on the large flanges can be safely extrapolated to determine the suitability of the freeze flange joint for a molten salt reactor system.

Approved b

APPENDIX 1

Preliminary Mechanical Joint Specification

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INTRA-LABORATORY CORRESPONDENCE OAK RIDGE NATIONAL LABORATORY

November 14, 1957

To: Listed Distribution

From: G. D. Whitman

Subject: Preliminary Mechanical Joint Specification

The attached specification has been prepared for your review and comment. This document is intended for internal use and should serve primarily as a design and development guide. Some of the values listed are more or less arbitrary; however, the criteria presented must be dealt with, and more considered limits may be assigned after a thorough review.

GDW/ds Att.

Distribution W. B. McDonald H. W. Savage H. G. MacPherson E. J. Breeding B. W. Kinyon L. A. Mann J. Zasler - 67 -

PRELIMINARY MECHANICAL JOINT SPECIFICATION

I. Scope

This specification sets forth the requirements for a mechanical joint which may be used in a radioactive high temperature molten salt system.

II. Service Requirements

This unit shall contain radioactive molten salts at temperatures up to 1500° F, and shall contain gaseous fission products between normal ambient and 1500° F. In addition, the unit shall be adaptable to remote assembly, disassembly and leak checking.

III. Materials of Construction

The materials of construction shall be compatible with the process fluids and in no way contribute contaminants which might reduce the life of the system or in any way reduce the efficiency of the nuclear or thermodynamic processes. Furthermore, it would be highly desirable to eliminate materials of construction which would result in high levels of induced activity.

IV. Mechanical Design

A. Size

The unit shall be adaptable to pipe sizes ranging from 1/2" IPS to 12" IPS.

B. Orientation

It is highly desirable that the unit be adaptable to piping runs of random orientation; however, a single application to vertical or horizontal runs shall be considered acceptable.

C. Holdup Volume

When the process fluid is drained from the system, the residual material, if any, shall not be more than 2 cc. Any material held up must be retained for inventory purposes and shall be handled in a manner that re-

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duces the spread of contamination.

D. Alignment Tolerance

The unit shall be capable of functioning to meet the requirements of this specification with a piping centerline mismatch of $\pm 1/32$ " and/or angularity misalignment between piping runs of $\pm 1/2^{\circ}$. These specifications are to be met without stressing the joining piping runs beyond their design limit.

E. Flow Resistance

The run of piping containing the unit shall not have any appreciable increase in flow resistance as compared to a similar section of regular pipe.

F. Strength

The unit shall behave the same as the parent piping under time, temperature, and stress and shall not require preferential application.

G. Remote Servicing

All special tools, fixtures, and leak checking equipment shall be considered as part of the unit package, and practical remote demonstration of such apparatus will be required.

H. General

Because of the remoteness of the application, it is desirable that auxiliary service requirements be kept to an absolute minimum.

V. Leak Tightness

- A. Gas Leakage
 - 1. Gas leaks shall be determined by means of a mass spectrometer leak detector having a sensitivity of 1×10^{-7} cc/sec. The system sensitivity shall be established before each test by calibration of the detector with a standard leak.

- 2. No indication of gas leakage shall be permitted. This leakage specification shall apply during and after 50 temperature cycles between ambient and 1200°F. The rate of temperature change shall not be less than 100°F per hour.
- Section V-A-2 of this specification is to be repeated after the unit has been remotely disassembled and assembled.

B. Fluid Leakage

- 1. No fluid leakage shall be permitted.
- 2. The unit shall be cycled 50 times between 1100°F and 1300°F with no indication of fluid leakage.
- 3. Section V-B-2 of this specification is to be repeated after the unit has been remotely disassembled and assembled.
- C. Special Leak Testing

Provision shall be made to establish the leak tightness of the unit during high temperature operation. Means shall be provided to collect the atmosphere surrounding the unit during high temperature fluid service to establish that gas contained in the fluid does not egress through the seal.

Prepared by: G. D. Whitman 11-11-57
APPENDIX 2

Graphs of Freeze Flange Thermal Cycling Tests

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APPENDIX 3

Graphs of Cast Seal Flange Thermal Cycling Tests

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CYCLE TIME (HOURS)

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CYCLE TIME (HOURS)

APPENDIX 4

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Graphs of Indented Seal Flange Thermal Cycling Tests





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APPENDIX 5

Report on Metallurgical Examination of the Cast Seal Flanges

SOLIDIFIED METAL SEAL DEVELOPMENT

G. M. Slaughter

In order to provide information to the designers of the solidified metal seal, several 0.252-in.-dia tensile bars were prepared from cast silver. These were tested at room temperature, $1200^{\circ}F$, and $1400^{\circ}F$. The test results indicate that the room-temperature mechanical properties are about the same as the published data; i.e., 20,000 psi tensile, 8,000 psi yield, and 50% elongation in 1-in. At $1200^{\circ}F$, the tensile strength dropped to 2,400 psi and the yield strength to 1,900 psi, with a corresponding elongation of 6 - 10%. At $1400^{\circ}F$, the tensile strength was 1,500 psi, the yield strength was 1,00 psi, and the elongation was 10%.

The two flanges that were operated in test at Y-12 were received for metallographic examination. A half-section of the silver flange is shown in Fig. 35 (Y-26029), while a half-section of the silver-copper flange is shown in Fig. 36 (Y-26103). The examination indicated that moderate oxidation of the components during opening and closing the flange had occurred, thereby impeding wetting of the base metal. The flange utilizing the silvercopper alloy appeared to be less subject to this condition than that containing the pure silver, probably because of the lower temperatures required to remelt it during opening. A photomicrograph of a typical interface between the silver-copper and the Inconel base metal is shown in Fig. 37 (Y-26241).



Fig. 35. Cast Seal Flange with Silver Seal, Sectioned.



Fig. 36. Cast Seal Flange with Copper-Silver Alloy Seal, Sectioned.

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Fig. 37. Cast Seal Flange with Copper–Silver Alloy Seal; Photomicrograph of Interface of Alloy and Inconel Wall.

APPENDIX 6

Report on Metallurgical Examination of the Freeze Flange Joint

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INTER-COMPANY CORRESPONDENCE

UNION CARBIDE NUCLEAR COMPANY A Division of Union Carbide and Carbon Corporation

то:	A. Taboada	Plant:	9201-3, Y-12
Copies to:	W. B. McDonald A. S. Olson RSC Files (FC)	Date:	May 26, 1958
		Subject:	Examination of Freeze Flange

This freeze flange had operated through several thermal cycles in test. Information regarding possible cracking was desired.

Three sections were taken through the center hole to explore all the areas likely to exhibit cracks. One section included a welded-on web and a longitudinal section through the center hole. Another section was taken roughly 45° away longitudinally through the hole and a third in transverse cross section through the counter-sunk center of the flange. It was in this area that cracking was felt to most likely occur.

None of the sections examined showed any cracks.

R. S. Crouse

RSC:fl
APPENDIX 7

Assembly Drawings of the Mechanical Joints and Loop

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- Freeze Flange Mechanical Joint 1/2" tubing size.
 Dwg. No. D-2-02-054-6777
- 2. Cast Seal Joint Dwg. No. C-2-02-054-6837
- Indented Seal Joint
 Dwg. No. D-2-02-054-7249
- 4. Freeze Flange Joint 4-inch pipe size Dwg. No. E-2-02-054-7014
- 5. Inconel Test Loop

Dwg. No. E-2-02-054-7126

(The drawing shows the loop configuration for freeze flange tests Nos. 3 and 4 and cast seal flange tests Nos. 1 and 2. For the indented seal flange tests, the cast seal flanges were cut out of the loop and replaced by lengths of 1/2 inch Incomel tubing. The freeze flanges were cut out of the loop and the indented seal flanges installed in their place.)



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		P/	ARTS LIST
	DWG. NO.	QTY.	NAME
1	D-6777-1	2	FLANGE SUB-ASSY.
2	0.6777-2	1	TUBE
3	D-6777-3	1	OVAL RING GASKET NO. R-40 SOFT STEEL
4	D-6777-4	B	CLAMP-LAPEER MANUE.



D-2-02-054-6777

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