Contract No. W-7405-eng-26

## REACTOR DIVISION

## ESTIMATED COST OF ADDING A THIRD SALT-CIRCULATING SYSTEM FOR CONTROLLING TRITIUM MIGRATION IN THE 1000-MW(e) MSBR

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JULY 1971

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#### Abstract

Controlling tritium migration to the steam system of the $1000-\mathrm{MW}(\mathrm{e})$ reference design MSBR power station by interposing a $\mathrm{KNO}_{3}-\mathrm{NaNO}_{2}-\mathrm{NaNO}_{3}$ salt-circulating system to chemically trap the tritium would add about $\$ 13$ million to the total of \$206 million now estimated as the cost of the reference plant if Hastelloy $N$ is used to contain the ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{2}$ salt employed to transport heat from the fuel salt to the nitrate-nitrite salt, and about $\$ 10$ million if Incoloy could be used. The major expenses associated with the modification are the costs of the additional heat exchangers ( $\$ 9$ million), the additional pumps ( $\$ 5$ million), and the ${ }^{7} \mathrm{LiF}-\mathrm{BeFa}$ inventory ( $\$ 4.8$ million). Some of the expense is offset by elimination of some equipment from the feedwater system ( $\$ 2$ million), through use of less expensive materials in the steam generators and reheaters (about $\$ 2$ million), and through an improved thermal efficiency of the plant (worth about $\$ 1$ million). In addition to acting as an effective tritium trap the third circulating system would make accidental mixing of the fuel and secondary salts of less consequence and would simplify startup and operation of the MSBR. A simplified flowsheet for the modified plant, a cell layout showing location of the new equipment, physical properties of the fluids, design data and cost estimates for the new and modified equipment are presented.


KEY WORDS $-{ }^{*}$ MSBR + *tritium + *capital cost + conceptual design + loop + coolants + heat exchangers + pumps + power costs + fuel-cycle costs + steam system.

## SUMMARY AND CONCLUSIONS

Controlling tritium migration to the steam system of the $1000-\mathrm{MW}(\mathrm{e})$ reference design MSBR power station by interposing salt-circulating loops to chemically trap the tritium would add 4 to $6 \%$ to the total plant cost. The net increase in capital cost of the plant, including indirect costs, is about $\$ 13$ million if Hastelloy $N$ is used to contain the ${ }^{7} \mathrm{LiFF}^{\mathrm{LiFeF}} \mathrm{a}_{\mathrm{a}}$ salt employed as the heat transport fluid in the secondary system, and about $\$ 10$ million if Incoloy could be used. These increases would apply to a cost for the reference design plant now estimated at about \$206 million (based on early 1970 costs). Addition of the loops would increase the power production costs by $0.2-0.3 \mathrm{mills} / \mathrm{kWhr}$, making the total cost about $5.5 \mathrm{mills} / \mathrm{kWhr}$.

As shown in the cost summary, Table l, the major portion of the cost of modifying the design is due to the additional heat exchangers and pumps required, and to the relatively high cost of the ${ }^{7}$ Li-bearing secondary salt. There were also increases in the cost of the primary heat exchangers and in the fuel-salt inventory. However, the added third loops use a nitrate-nitrite heat transport salt which permits savings in the material costs in the steam generators and reheaters. Use of this salt also permits reductions in the feedwater and cold reheat steam temperatures, and through changes in the steam system flowsheet and the auxiliary electric load, produces a reduction of costs equivalent to a plant investment of about $\$ 800,000$. Credit for these savings was taken in the net costs mentioned above.

In addition to serving as an effective tritium trap, the third loops offer other important advantages over the reference design. These are features which, in general, could not have cost credits assigned. For example, the similarity of the fuel and secondary salts makes mixing due to leaks in the primary heat exchanger of far less consequence than in the reference design. Startup and operation of the MSBR would be simplified because of changes that could be made in the steam system flowsheet.

Table l. Summary of Cost Items Affected by Modifying MSBR Reference Design to Include Third Salt-Circulating Loops (in \$1000)

| Rev. Reference <br> Design MSBR |
| :---: |
| Modified MSBR <br> with Third Ioops |

A. With Hastelloy N secondary system

Revised equipment:

| Primary heat exchangers (see Table 4) | $\$ 8,660$ | $\$ 9,880$ |
| :--- | ---: | ---: |
| Steam generators (see Table 6) | 7,230 | 6,192 |
| Steam reheaters (see Table 7) | 1,565 | 1,216 |
| Coolant salt pumps (see Table 1l) | 4,400 | 2,750 |
| Coolant salt piping allowance | 1,900 | 1,500 |
| Coolant salt drain tank | 800 | 800 |
| Coolant salt inventory cost | 500 | 135 |
| Auxiliary boiler allowance | 3,000 | 2,500 |

New equipment:
Secondary heat exchanger (see Table 5)
6,883
Secondary pumps (see Table ll)
3,800
Secondary salt drain tank
800
Secondary system piping allowance
375
Accessory electrical for secondary 200 system
Eliminated equipment:
Reheat steam preheaters (see Table 8) 1,056
Pressure-booster pumps 650
Mixing chambers
Total direct construction cost, in \$1000 \$29,841
Difference in direct construction costs
\$7,190
Difference in total cost with added \$9,563
indirect costs of $33 \%$
${ }^{7}$ LiF- ${ }^{2} \mathrm{BeF}$ a inventory cost (see Tables 13 and 14)

Credit for resale value of ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{a}$
4,800

Credit for improved plant efficiency
$-239$
(see Table 12)
Net estimated capital cost of adding third
\$13,300 loops

Table 1 (continued)

|  | Rev. Reference Design MSBR | Modified MSBR with Third Loops |
| :---: | :---: | :---: |
| Changes in power production cost: |  | $\underline{\mathrm{mill}} / \mathrm{kWhr}$ |
| Net cost of adding third loops, at $13.7 \% \mathrm{FC}$ | 7\% FC +0 |  |
| LiF- $\mathrm{BeF}_{\text {a }}$ inventory, at $13.2 \% \mathrm{FC}$ |  | . 090 |
| Credit for resale $\mathrm{LiF}-\mathrm{BeF}_{3}$, at $13.2 \% \mathrm{FC}$ |  | . 005 |
| Credit for improved efficiency, at $13.7 \% \mathrm{FC}$ | 7\% FC - 0 | . 015 |
| Increase in fuel-cycle cost |  | . 013 |
| Net increase in cost of power |  | 27 mills/kWhr |
| B. With Incoloy secondary system |  |  |
| All items in modified MSBR not affected by use of Incoloy rather than Hastelloy $N$ in secondary circulating loop, from Part A, above. | above. | 9,893 |
| Cost of items in which Incoloy is substituted for Hastelloy N: |  |  |
| Primary heat exchangers, (see Table 4) |  | 8,661 |
| Secondary salt piping allowance |  | 225 |
| Secondary heat exchangers (see Table 5) |  | 5,879 |
| Cost of revised reference design, from Part A | At \$ | $\begin{aligned} & 4,658 \\ & 9,841 \end{aligned}$ |
| Difference in direct construction costs |  | 4,817 |
| Difference in total cost with indirect costs of $33 \%$ added |  | 6,407 |
| ${ }^{7} \mathrm{LiF-BeF} \mathrm{a}_{\mathbf{a}}$ inventory cost (see Tables 13 and 14) | and 14) | 4,800 |
| Credit for resale value of ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{\mathbf{z}}$ |  | -239 |
| Credit for improved plant efficiency (see Table 12) | Table 12) | -817 |
| Net estimated cost of adding third loops |  | ,200 |
| Changes in power production cost | mills | /kWhr |
| Net cost adding third loops, at $13.7 \%$ FC | $\mathrm{FC}+0$ | . 125 |
| LiF- $\mathrm{BeF}_{\text {a }}$ inventory, at $13.2 \% \mathrm{FC}$ | + 0 | . 090 |
| Credit for resale LiF-BeF ${ }_{3}$, at $13.2 \% \mathrm{FC}$ | C -0. | . 005 |
| Credit for improved efficiency, at $13.7 \%$ FC | . $\% \mathrm{FC}-0$ | . 015 |
| Increase in fuel-cycle cost | +0. | . 013 |
| Net increase in cost of power. | + 0 | . 21 mills/kWhr |

## 1. INIRODUCTION

Tritium formed in the MSBR fuel salt must be prevented from reaching the steam system. The problem is difficult because of the relative ease with which hydrogen diffuses through most metals at MSBR operating temperatures. Studies are being made at ORNL of several different methods of tritium control; of these, the introduction of a third salt-circulating system to chemically trap the tritium between the secondary salt and the steam system is the only one well within present technology and, on the basis of present knowledge, offers assured confinement of the tritium. It is possibly one of the most expensive of the control methods being considered, however, and raises the question as to whether its use would add prohibitively to the cost of a molten-salt reactor power station.

This study evaluates the various cost factors involved in adding the third salt-circulating system to the $1000-\mathrm{MW}(\mathrm{e}) \mathrm{MSBR}$ reference design described in ORNL-454.1. The cost estimating methods follows those used in that report. The costs of modifying the reference design include the capital cost of the extra equipment, the salt inventories, and also reflect the cost effects of the new designs for the heat transfer equipment made necessary by the use of heat transfer fluids different from those used in the reference concept. (The calculations for the new and modified heat exchangers were made by C. E. Bettis et al., using essentially the same computer programs as were used in the reference design.) The cost estimates also take credit for the equipment not needed in the feedwater system of the modified plant and for the improved thermal efficiency of the station, as explained below.

The reference MSBR design uses circulating sodium fluoroborate, $\mathrm{NaF}-\mathrm{NaBF}_{4}$, to transport heat to the steam generators and reheaters, whereas the modified design uses a nitrate-nitrite heat transfer salt, $\mathrm{KNO}_{3}-\mathrm{NaNO}_{2}$ $\mathrm{NaNO}_{3}$ (known commercially as "Hitec"), to heat the steam equipment. This has five important advantages: (1) any hydrogen diffusing into the salt

[^0]would combine with the oxygen and subsequently be drawn off as steam and collected, forming an effective tritium trap; (2) the salt is not corrosive to less expensive materials of construction, allowing Incoloy 800, or a similar material, to be substituted for the Hastelloy N used in the reference design; (3) its low melting temperature of $288^{\circ} \mathrm{F}$ permits use of conventional feedwater and cold reheat temperatures in the steam system and eliminates the need for the reheat steam preheaters, the pressurebooster pumps and mixing chambers used in the reference design; (4) startup of the system is simplified and the auxiliary boiler probably does not need to be a supercritical-pressure unit as in the reference plant; and (5) the salt has a low cost of only about 15 cents/lb. The salt does not react exothermically with water and it has good flow and heat transfer properties.

The modified design would use a ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{2}$ salt to transport heat from the fuel salt to the nitrate-nitrite salt. With the exception of the uranium and thorium components, this salt is the same as the fuel salt, and thus a leak in the primary heat exchanger would be of far less consequence than in the reference design where dissimilar salts would mix. The ${ }^{7}$ LiF-BeF ${ }_{a}$ is not corrosive to materials less expensive than Hastelloy $N$, provided that no moisture is present. One cost estimate in this study has been made using Hastelloy $N$ for the secondary system and another using Incoloy. Due to the lithium-7 content, the cost of the salt is relatively high -- about $\$ 12 / \mathrm{lb}$. Its resale value at the end of the 30-year plant life has been taken into account, although the effect is not great.

The reference MSBR design consists of a single reactor supplying heat to four primary circulating loops, each containing a salt-circulating pump and a heat exchanger. The coolant-salt system contains four loops, with each containing a salt-circulating pump, four steam generators and two reheaters. This arrangement was not altered in the modified design, although there was some adjustment of the temperatures. The interposed salt-circulating system would consist of four loops, each containing a circulating pump and a heat exchanger. The following terminology has been adopted.

Fuel salt to ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{z}$ heat exchanger -- Primary heat exchanger
$\mathrm{LiF}-\mathrm{BeF}_{3}$ to $\mathrm{KNO}_{3}-\mathrm{NaNO}_{3}-\mathrm{NaNO}_{3}$ exchanger -- Secondary heat exchanger
$\mathrm{KNO}_{3}-\mathrm{NaNO}_{3}-\mathrm{NaNO}_{3}$ to steam exchangers -- Steam generator or steam reheater

Fuel-salt circulating pump -- Primary pump
LiF- $\mathrm{BeF}_{2}$ circulating pump -- Secondary pump
$\mathrm{KINO}_{3}-\mathrm{NaNO}_{2}-\mathrm{NaNO}_{3}$ circulating pump -- Tertiary pump
This study is primarily concerned with evaluating the cost effects of adding the third salt-circulating loops. The concept was not carried further than to indicate general feasibility and to provide a basis for cost estimates. No effort was made toward optimization.

In comparing the cost of the MSBR modified with the third loops to the reference design cost estimates, it was necessary to make some revisions to the latter as reported in ORNL-4541. The heat transfer equipment design data have undergone two relatively recent revisions. The first was made in time to be tabulated with the design data in the latest distributed draft of the report, but, because of the extensive changes required and the fact that at the time the influence on costs appeared to be small, the cost estimates were not adjusted accordingly. The second revision, which applied only to the primary heat exchanger, was made just in time for the data to be changed before the report was printed, but, again, the cost estimates could not be revised. All of the revisions tended to increase costs, however, and when the cost estimates were revised in this study it was found that in aggregate they amounted to about $\$ 4$ million, including the indirect charges. The total capital cost of the reference design MSBR is thus about $\$ 206$ million rather than the $\$ 202$ million given in ORNL-4541. Both amounts are based on the early 1970 value of the dollar.
2. DESCRIPTION OF MSBR MODIFIED WITH THIRD LOOPS

A simplified flowsheet for the $1000-\mathrm{MW}(\mathrm{e})$ MSBR station as modified to include the third salt-circulating loops is shown in Fig. I. It can be noted that the temperatures have been adjusted from those used in the reference design and that there were corresponding changes in the mass


Fig. l. Schematic Flowsheet of 1000-MW(e) MSBR Power Station as Modified with Addition of Third Loops to Trap ${ }^{3} \mathrm{H}$.
flow rates of the salts. The flow quantities shown on the flowsheet are for each of the four circulating loops.

The secondary heat exchangers and the associated $\mathrm{LiF}^{-}-\mathrm{BeF}_{\mathbf{a}}$ pumps can be arranged in the reactor cell without changing the dimensions of the containment structure, as indicated in Fig. 2. The layout provides relatively short piping between the primary and secondary heat exchangers to keep the lithium-7 inventory low. No major changes would be required in the salt piping to the steam generators and reheaters. On this basis, the cost estimates for the modified system do not include any expenses for modification of the building or cell structure.

## 3. HEAT TRANSFER EQUIPMENT

The physical properties of interest for the fuel and heat-transport salts are given in Table 2. (Sodium fluoroborate has been included for comparison, although not used in the modified MSBR system.)

The costs of the heat transfer equipment were based on the estimated weights of the various shapes of materials used in fabrication, and on a unit price which reflects the costs of fabrication, inspection, transportation, and installation ready for use. The total installed costs of Hastelloy N and Incoloy 800, as used in this study, are listed in Table 3. As in the reference design, the base prices of materials can be determined with relatively good certainty, but the additions to provide the total installed cost greatly overshadow the basic material cost in importance and also involve considerable intuitive judgment. As a rough check on the reasonableness of the cost estimates, the costs per square foot of heat transfer surface are compared in Table 10.

## 1. Primary Heat Exchangers

The cost estimate for the primary heat exchangers in the reference design, as reported in ORNL-4541, has been changed from $\$ 7.3$ million to about $\$ 8.7$ million to reflect the revisions to the design data, as indicated in Table 4. The cost increase is also due to adding in the cost of the baffles and to inclusion of the double-pipe coolant-salt nozzles, which had previously been assumed to be covered by the piping cost


Fig. 2. MSER Reactor Cell Layout Indicating Possible Location for Secondary Heat Exchanger and Pump. (One of four loops is shown.)

Table 2. Selected Properties of the MSBR Molten Salts

|  | ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{2}-\mathrm{ThF}_{4}-\mathrm{UF}_{4}$ | $\mathrm{NaF}-\mathrm{NaBF}_{4}$ | ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{2}$ | $\mathrm{KNO}_{3}-\mathrm{NaNO}_{2}-\mathrm{NaNO}_{3}$ |
| :---: | :---: | :---: | :---: | :---: |
| Composition, mole \% | 71.7-16-12-0.3 | 92-8 | 66-34 | 44.2-48.9-6.9 ${ }^{\text {a }}$ |
| Molecular weight, approximate | 64 | 104 | 33 | 84 |
| Density, $1 \mathrm{~b} / \mathrm{ft}^{3}$ at $1000^{\circ} \mathrm{F}$ | 212 | 117 | 124 | 105 |
| Viscosity, $1 \mathrm{l} / \mathrm{ft}-\mathrm{hr}$ at $1000^{\circ} \mathrm{F}$ | 41 | 3 | 29 | 3 |
| Specific heat, $\mathrm{Btu} / \mathrm{Ib}-{ }^{\circ} \mathrm{F}$ | 0.32 | 0.36 | 0.57 | 0.37 |
| Thermal conductivity, Btu/ft-hr- ${ }^{\circ} \mathrm{F}$ | 0.67 to 0.68 | 0.23 | 0.58 | 0.33 |
| Estimated cost, \$/Ib | 57.00 | 0.50 | 12.00 | 0.15 |
| Circulation required per loop ${ }^{\text {b }}$ for 556-MW(t) heat load: |  |  |  |  |
| 1b/hr | $23.4 \times 10^{6}$ | $18.3 \times 10^{6}$ | $13.3 \times 10^{6}$ | $14.7 \times 10^{6}$ |
| gpm | 14,260 | 19,500 ${ }^{\text {c }}$ | 13,380 | 17,370 |
| Liquidus temperature, ${ }^{\circ} \mathrm{F}$ | 930 | 725 | 850 | 288 |

${ }^{\text {a }}$ Eutectic composition.
${ }^{\mathrm{b}}$ Based on properties at average temperatures in MSBR system.
${ }^{\mathrm{C}}$ Based on $250^{\circ} \mathrm{F} \Delta t$ in modified MSBR.

Table 3. Material Costs Used in Estimates ${ }^{\text {a }}$

|  | Hastelloy N | Incoloy |
| :--- | :---: | :---: |
| Tubes, $3 / 8$ in. diam | $\$ 30 / 1 \mathrm{~b}$ | $\$ 28 / 1 \mathrm{~b}$ |
| $1 / 2$ in. diam and larger | 20 | 17 |
| Shells and liners | 10 | 7 |
| Heads | 15 | 12 |
| Baffles | 15 | 12 |
| Rings | 20 | 18 |
| Tubesheets | 20 | 12 |
| Downcomers, large nozzles | 15 | 18 |

${ }^{a}$ Includes cost of material, fabrication, transportation, inspection, and installation ready for use.
allowance. It was also found that the inside diameter of the shell stated in ORNL-4541 appiied to the inner liner rather than to the outer shell.

The design data for the primary heat exchangers as modified to use LiF-BeFa on the shell side are also shown in Table 4. These design data have not been recalculated using the May 1971 revisions to the computer program (see Introduction), but the effects of the changes could be estimated by using their influence on the reference design primary heat exchanger costs as a guide, as follows: tubes ( $+6.4 \%$ ), shell and liner ( $+8.7 \%$ ), heads ( $-1.4 \%$ ), rings ( $-1.0 \%$ ), downcomers, U-bends and baffles ( $+4.1 \%$ )

The tubes and other portions of the primary heat exchanger in contact with the fuel salt must be constructed of Hastelloy $N$. This was also true in the reference design for the portions in contact with the sodium fluoroborate salt. In the modified design, however, consideration

Table 4. Primary Heat Exchangers

|  | Revised Reference Design MS.BR | Modified MSBR With Third Loop |
| :---: | :---: | :---: |
| Capacity, MW( $t$, each of four units | 556 | 556 |
| Fuel salt temperatures, in-out, ${ }^{\circ} \mathrm{F}$ | 1300-1050 | 1300-1050 |
| Coolant salt temperature, in-out, ${ }^{\circ} \mathrm{F}$ | 850-1150 | 950-1200 |
| Coolant salt | $\mathrm{NaF}-\mathrm{NaBF}_{4}$ | $\mathrm{LiF}-\mathrm{BeF}_{2}$ |
| Tube size (enhanced), OD $\times$ wall thickness, in. | $3 / 8 \times 0.035$ | $3 / 8 \times 0.035$ |
| Number of tubes | 5803 | 6312 |
| Length of tubes, ft | 24.4 | 25.5 |
| Heat transfer area, fta | 13,916 | 15,789 |
| Liner, ID x thickness, in. | $67.6 \times 2.5$ | $70.3 \times 2.5$ |
| Shell, ID $x$ thickness, in. | $73.6 \times 1 / 2$ | $76.3 \times 1 / 2$ |
| Pressure drops: tube side, psi | 130 | 130 |
| shell side, psi | 116 | 118 |
| Head thickness, in. | 3/4 | 3/4 |
| Number of baffles, disc and doughnut, $3 / 8$ in. thick | 21 | 34 |
| Overall heat transfer coefficient, $\mathrm{Btu} / \mathrm{hr}-\mathrm{ft}^{2}-{ }^{\circ} \mathrm{F}$ | 785 | $672-944$ |

A. Material costs with Hastelloy $N$ tubes and shell (in $\$ 1000$ ):

| Tubes, at $\$ 30 / 1 \mathrm{~b}$ | $\$ 2,457$ | $\$ 2,970$ |
| :--- | ---: | ---: |
| Shells, at $\$ 10 / 1 \mathrm{~b}$ | 414 | 487 |
| Liners, at $\$ 10 / 1 \mathrm{~b}$ | 1,959 | 2,308 |
| Heads, at $\$ 15 / 1 \mathrm{~b}$ | 141 | 150 |
| Rings and tube sheets, at $\$ 20 / 1 \mathrm{~b}$ 2,823 |  |  |
| Downcomers, baffles, and double- <br> pipe coolant nozzles, at $\$ 15 / 1 \mathrm{~b}$ <br> Installation allowance | 666 | 8,911 |
| $\quad$ Total for four units | $\$ 8,660$ | $\$ 9,880$ |

(continued)

Table 4 (continued)

| Revised Reference Design MSBR | Modified MSBR With Third Loop |
| :---: | :---: |
| B. Material costs with Hastelloy $\mathbb{N}$ tubes and Incoloy sh | 1 (in \$1000): |
| Tubes, at \$30/ıb | \$ 2,970 |
| Shells, at \$8/Ib | 350 |
| Liners, at \$8/ıb | 1,658 |
| Heads, at \$15/工b | 150 |
| Hastelloy $\mathbb{N}$ rings and tubesheets, at \$20/1b | 1,907 |
| Incoloy rings, at \$17/1b | 812 |
| Downcomer, at \$12/lb | 126 |
| Double-pipe coolant nozzles, at $\$ 12 / \mathrm{lb}$ | 65 |
| Baffles, at \$12/lb | 424 |
| Installation allowance | 200 |
| Total | \$8,661 |

can be given to use of less expensive materials in the shell side of the system, provided that no moisture is present. The more conservative approach is to use Hastelloy $\mathbb{N}$ for all portions of the secondary system, and this is the basis for the cost estimates shown in Part A of Tables 1, 4, and 5. Since there has been noteworthy success in excluding water from salt systems, however, it may be practical to use Incoloy, or a similar material, in the secondary system. The estimated costs in this case are shown in Part B of Tables 1, 4, and 5. It will be noted that use of Incoloy would save about $\$ 3$ million in total costs when indirect charges are included.

## 2. Secondary Heat Exchangers

The secondary heat exchangers in the modified MSBR plant are envisioned as U-shell and U-tube types, arranged vertically in the reactor cell, as indicated in Fig. 2. The design data were generated on the basis of four units with $3 / 8$-in.-OD tubing. The arrangement was not

Table 5. Secondary Heat Exchangers

|  | Modified MSBR With Third Loop |
| :---: | :---: |
| Capacity, each of four units, MW( $t$ ) | 556 |
| $\mathrm{LiF}-\mathrm{BeF}_{z}$ (tubes) temperatures, in-out, ${ }^{\circ} \mathrm{F}$ | 1200-950 |
| $\mathrm{KNO}_{3}-\mathrm{NaNO}_{2}-\mathrm{NaNO}_{3}$ (shell) temperatures, in-out, ${ }^{\circ} \mathrm{F}$ | 750-1100 |
| Tube size (not enhanced), OD $\times$ wall thickness, in. | $3 / 8 \times 0.035$ |
| Number of tubes | 5989 |
| Length of tubes, ft | 44 |
| Heat transfer surface, $\mathrm{ft}^{\text {a }}$ | 25,665 |
| Pressure drops: tube side, psi shell side, psi | $\begin{aligned} & 79.2 \\ & 79.6 \end{aligned}$ |
| Shell, ID $\times$ wall thickness, in. | $61.5 \times 1 / 2$ |
| Number of baffles, crosscut, $3 / 8 \mathrm{in}$. thick | 33 |
| Tubesheet thickness, in. | 3 |
| Head thickness, in. | 3/4 |
| Overall heat transfer coefficient, Btu/hr-ft ${ }^{\text {- }}{ }^{\circ} \mathrm{F}$ | 505 |
| A. Material cost with Hastelloy $\mathbb{N}$ tubes and Incoloy shell (in \$1000): |  |
| Tubes, at \$30/1b | \$ 4, 542 |
| Shell, at \$8/lb | 483 |
| Thubesheet, at \$20/lb | 458 |
| Heads, at \$15/lb | 102 |
| Baffles, at \$12/lb | 1,018 |
| Nozzles, etc., at \$20/lb | 80 |
| Installation allowance | 200 |
| Total for four units | \$ 6,883 |
| B. Material cost with Incoloy shell and tubes (in \$1000) : |  |
| Tubes, at \$27/Ib | \$ 3,670 |
| Shell, at $\$ 8 / \mathrm{lb}$ | 483 |
| Tubesheets, at \$18/1b | 370 |
| Heads, at \$12/1b | 78 |
| Baffles, at \$12/1b | 1,018 |
| Nozzles, etc., at \$18/1b | 65 |
| Installation allowance | 200 |
| Total for four units | \$ 5,879 |

optimized, however, and although sufficient for cost-estimating purposes, there are indications that further study may be needed. For example, the calculated shell diameter of over 60 in . is questionable for the Ushell configuration. The tube size needs optimizing in that the $3 / 8$-in.OD tubing is needed to minimize the $\mathrm{LiF}_{\mathrm{Be}} \mathrm{BeF}_{2}$ inventory and surface requirements, but it is relatively expensive compared to larger sizes (see Table 3). Consideration could be given to use of eight units rather than four, and to use of straight-tube designs, although space in the cell is somewhat limited.

As previously discussed, there is a possible option in selecting materials to be used on contact with the $\mathrm{LiF}-\mathrm{BeF}_{2}$ salt. Part A of Table 5 shows the estimated direct cost of the secondary heat exchangers if constructed with Hastelloy $\mathbb{N}$ tubes and heads, and Part B indicates the cost if Incoloy is used for these parts.

## 3. Steam Generators

The cost estimate for the steam generators in the reference design was changed from $\$ 6.3$ million to $\$ 7.2$ million to reflect the revisions in the design data. The principal differences were due to an increase in the number and length of the tubes and an increase in the thickness of the tube sheets used in the cost estimate. The data and costs are shown in Table 6.

The design data and the estimated cost of the steam generators for the modified MSBR system using $\mathrm{KNO}_{3}-\mathrm{NaNO}_{3}-\mathrm{NaNO}_{3}$ on the shell side are also shown in Table 6. The lower total cost of the units for the modified design is primarily due to use of Incoloy rather than Hastelloy N. It may be noted that the steam generators are designed for $555^{\circ} \mathrm{F}$ entering feedwater rather than the $551^{\circ} \mathrm{F}$ temperature called for in the flowsheets. A technicality in the computer program made it necessary to revise the number, but since the total amount of heat to be transferred was not altered, the only sacrifice to accuracy was relatively small velocity effects.

Table 6. Steam Generators

|  | Revised Ref'erence Design MSBR | Modified MSBR With Thira Loop |
| :---: | :---: | :---: |
| For each of 16 units: |  |  |
| Capacity, MW(t) | 121 | 121 |
| Type | U-shell, U-tube | U-shell, U-tube |
| Major material of construction | Hastelloy N | Incoloy 800 |
| Heat transport salt (shell side) | $\mathrm{NaF}-\mathrm{Na}_{3} \mathrm{Br}_{4}$ | $\mathrm{KNO}_{3}-\mathrm{NaNO}_{3}-\mathrm{NaNO}_{3}$ |
| Salt temperatures, in-out, ${ }^{\circ} \mathrm{F}$ | 1150-850 | 1100-750 |
| Feedwater temperature, ${ }^{\circ} \mathrm{F}$ | 700 | 555 |
| Steam temperature out, ${ }^{\circ} \mathrm{F}$ | 1000 | 1000 |
| Steam pressure, psia | 3625 | 3625 |
| Tube size, $O D \times$ wall thickness, in. | $1 / 2 \times 0.077$ | $1 / 2 \times 0.077$ |
| Number of tubes | 393 | 341 |
| Tube length, ft | 76 | 99 |
| Heat transfer surface, $\mathrm{ft}^{\text {a }}$ | 3929 | 4428 |
| Shell, ID $x$ wall thickness, in. | $18.3 \times 3 / 8$ | $17 \times 3 / 8$ |
| Number of baffles ( $3 / 8 \mathrm{in}$. thick) | 18 | 28 |
| Head (spherical) thickness, in. | 4 | 4 |
| Pressure drops: tube side, psi shell (salt) side, psi | $\begin{array}{r} 152 \\ 61 \end{array}$ | $\begin{array}{r} 125 \\ 90 \end{array}$ |
| Overall U, Btu/ $\mathrm{hr}-\mathrm{ft}^{2}{ }^{\circ} \mathrm{F}$ | - | 655 |
| Material costs (all 16 units) |  |  |
| Tubes: Cost, $\$ / \mathrm{lb}$ <br> Total cost (\$1000) | $\begin{gathered} (20) \\ \$ 3,803 \end{gathered}$ | $\begin{gathered} (17) \\ \$ 3,269 \end{gathered}$ |
| Shells: Cost, $\$ / 1 \mathrm{~b}$ <br> Total cost (\$1000) | $\begin{array}{r} (10) \\ 1,046 \end{array}$ | $\begin{aligned} & (8) \\ & 910 \end{aligned}$ |
| $\begin{array}{ll} \text { Heads: } & \text { Cost, } \$ / 1 \mathrm{~b} \\ & \text { Total cost ( } \$ 1000 \text { ) } \end{array}$ | $\begin{aligned} & (15) \\ & 565 \end{aligned}$ | $\begin{aligned} & (12) \\ & 406 \end{aligned}$ |
| Tubesheets: Cost, $\$ / \mathrm{lb}$ Total cost (\$1000) | $\begin{array}{r} (20) \\ 1,016 \end{array}$ | $\begin{aligned} & (18) \\ & 821 \end{aligned}$ |
| Misc.: Cost, $\$ / \mathrm{lb}$ <br> Total cost (\$1000) | $\begin{aligned} & (20) \\ & 320 \end{aligned}$ | $\begin{aligned} & (18) \\ & 306 \end{aligned}$ |
| Installation allowance | 480 | 480 |
| Total cost (\$1000) | \$ 7,230 | \$6,192 |

## 4. Steam Reheaters

The estimated cost of the steam heaters in the reference design was revised from $\$ 1.7$ million to $\$ 1.6$ million to correspond to the revised design data, as shown in Table 7. Although the revised unit has more surface, the previously used price of Hastelloy $\mathbb{N}$ tubing did not reflect the lower unit price of 3/4-in.-OD tubing as compared to 3/8-in.-OD tubing.

The design data and estimated cost of the modified reheaters using $\mathrm{KNO}_{3}-\mathrm{NaNO}_{2}-\mathrm{NaNO}_{3}$ on the shell side are also shown in Table 7. Using Incoloy rather than Hastelloy $N$ accounted for the reduction in cost to $\$ 1.2$ million. It will be noted that the unit is designed for $550^{\circ} \mathrm{F}$ entering cold reheat temperature, as taken directly from the high-pressure turbine exhaust.

## 5. Reheat Steam Preheaters

Reheat steam preheaters were used in the reference design to heat the high-pressure turbine exhaust from $550^{\circ} \mathrm{F}$ to $650^{\circ} \mathrm{F}$ before the steam entered the reheaters to avoid possible problems of coolant-salt freezing. The cost of the preheaters was underestimated in the reference design report because the thickness of the spherical heads was used in the calculations as $1 / 2$-in. rather than the correct value of $2-1 / 2$ in. Further, the material costs assumed for the Croloy in the reference design appeared too low. The revised cost estimate for the preheaters is now $\$ 924,000$, as shown in Table 8.

The preheater design was not optimized. Use of $3 / 8$-in. tubes may involve a cost penalty, and some improvement in costs might be obtained if the number of units was increased.

The modified MSBR with the third loops added to trap tritium does not require use of preheaters because of the low liquidus temperature of the nitrate-nitrite salt.

## 6. General Effects of Revising and Modifying the Heat Transfer <br> Equipment

The total cost effects of revising the design data for the heat transfer equipment in the reference design are summarized in Table 9 . The net increase of about $\$ 4$ million (including indirect charges)

Table 7. Steam Reheaters

|  | Revised Reference Design MSBR | Modified MSBR With Third Loop |
| :---: | :---: | :---: |
| For each of 8 units: |  |  |
| Capacity, MW(t) | 36.6 | 36.6 |
| Major material of construction | Hastelloy N | Incoloy 800 |
| Heat transport salt | $\mathrm{NaF}-\mathrm{NaBF}_{4}$ | $\mathrm{KNO}_{3}-\mathrm{NaNO}_{2}-\mathrm{NaNO}_{3}$ |
| Salt temperatures, in-out, ${ }^{\circ} \mathrm{F}$ | 1150-850 | 1100-750 |
| Steam temperature in, ${ }^{\circ} \mathrm{F}$ | 650 | 550 |
| Steam temperature out, ${ }^{\circ} \mathrm{F}$ | 1000 | 1000 |
| Entrance steam pressure, psia | 580 | 580 |
| Tube size, $O D \times$ wall thickness, in. | $3 / 4 \times 0.035$ | $3 / 4 \times 0.035$ |
| Number of tubes | 400 | 696 |
| Tube length | 30 | 28 |
| Heat transfer surface, $\mathrm{ft}^{\text {2 }}$ | 2376 | 2520 |
| Shell, ID x wall thickness, in. | $21.2 \times 0.5$ | $21 \times 0.5$ |
| Number of disc and doughnut baffles | 21 \& 21 | $30 \& 29$ |
| Head thickness, in. | 0.5 | 0.5 |
| Pressure drops: tube side, psi shell side, psi | $\begin{aligned} & 30 \\ & 60 \end{aligned}$ | $\begin{aligned} & 40 \\ & 90 \end{aligned}$ |
| Overall U, Btu/hr-ft ${ }^{\text {a }}{ }^{\circ} \mathrm{F}$ | 306 | 340 |
| Material costs (all 8 units): |  |  |
| Tubes: $\quad \$ / \mathrm{lb}$ cost, in \$1000 | $\begin{array}{ll} \$ \quad(20) \\ 590 \end{array}$ | $\begin{aligned} & (17) \\ & 465 \end{aligned}$ |
| Shells: $\quad \begin{aligned} & \$ / 1 b \\ & \text { cost, in } \$ 1000\end{aligned}$ | $\begin{aligned} & (10) \\ & 327 \end{aligned}$ | $\begin{gathered} (8) \\ 210 \end{gathered}$ |
| Tubesheets: $\begin{aligned} & \$ / 1 \mathrm{~b} \\ & \text { cost, in } \$ 1000\end{aligned}$ | $\begin{aligned} & (20) \\ & 146 \end{aligned}$ | $\begin{aligned} & (18) \\ & 115 \end{aligned}$ |
| Heads: $\quad \begin{aligned} & \$ / \text { lb } \\ & \text { cost, in } \$ 1000\end{aligned}$ | $\begin{gathered} (1.5) \\ 72 \end{gathered}$ | (12) |
| Baffles: $\quad \begin{aligned} & \$ / \text { Ib } \\ & \text { cost, in } \$ 1000\end{aligned}$ | $\begin{aligned} & (15) \\ & 151 \end{aligned}$ | $\begin{aligned} & (12) \\ & 109 \end{aligned}$ |
| Nozzles, etc: $\$ /$ Ib cost, in \$1000 | $\begin{gathered} (20) \\ 80 \end{gathered}$ | $\frac{(18)}{65}$ |
| Installation allowance | 200 | 200 |
| Total cost, in \$1000 | \$ 1,566 | \$ 1,216 |

## Table 8. Reheat Steam Preheaters

|  | Revised Reference Design MSBR |
| :---: | :---: |
| For each of 8 units: |  |
| Capacity, MW(t) | 12.3 |
| Major material of construction | Croloy |
| Shell-side conditions: |  |
| Heated steam entrance temperature, ${ }^{\circ} \mathrm{F}$ | 551 |
| Entrance pressure, psia: | 595 |
| Tube-side conditions: |  |
| Heating steam entrance temperature, ${ }^{\circ} \mathrm{F}$ | 1000 |
| Entrance pressure, psia | 3600 |
| Tube size, $O D \times$ wall thickness, in. | $3 / 8 \times 0.065$ |
| Number of tubes | 603 |
| Tube Iength, ft | 13.2 |
| Heat transfer surface, $\mathrm{ft}^{\text {a }}$ | 781 |
| Shell, ID $\times$ wall thickness, in. | $20-1 / 4 \times 7 / 16$ |
| Overall U, Btu/hr-ft ${ }^{\circ}{ }^{\circ} \mathrm{F}$ | 162 |
| Head thickness, in. | 2-1/2 |
| Material costs (all 8 units), in \$1000: |  |
| Tubes, at $\$ 18 / \mathrm{lb}$ | \$ 252 |
| Shells, at \$8/1b | 88 |
| Heads, at \$10/Ib | 2.96 |
| Tubesheets, at \$18/lb | 323 |
| Nozzles, etc., at \$18/1b | 72 |
| Installation allowance | 25 |
|  | \$ 1,056 |

Table 9. Revised Reference Design Costs for Heat Transfer Equipment (in $\$ 1000$ )

${ }^{\mathrm{a}}$ As listed in ORNL-4541.
${ }^{\mathrm{b}}$ For Hastelloy $\mathbb{N}$ fuel and coolant-salt systems.
results in raising the total estimated plant cost of the reference design MSBR plant from about $\$ 202$ million to $\$ 206$ million.

Use of Incoloy rather than Hastelloy $\mathbb{N}$ for the portions of the secondary system in contact with LiF- $\mathrm{BeF}_{8}$ would save about $\$ 1.6$ million in the total cost (including indirect charges) of the primary heat exchangers, about $\$ 1.3$ million for the secondary heat exchangers, and about $\$ 200,000$ for the secondary salt piping, for a total savings of about $\$ 3$ million.

The costs of the heat transfer equipment on a square foot basis are compared in Table $\mathbf{L O}$. While the values are not particularly conclusive, they indicate that the estimated costs are generally within reason for this type of nuclear power station equipment.

## 4. SALT-CIRCULATING PUMPS

Since salt-circulating pumps of the size required for the $1000-\mathrm{MW}(\mathrm{e})$ MSBR station have never been fabricated, the cost-estimating method used

Table 10. Estimated Direct Cost of Installed Heat Transfer
Equipment per Square Foot of Surface

Revised Reference Design MSBR

Modified MSBR
With Third Loop

Primary heat exchangers
Hastelloy N tubes and shell
\$ 1.55
$\$ 147$
Hastelloy N tubes and Incoloy shell
-
129
Secondary heat exchangers
Hastelloy $N$ tubes and shell67

Hastelloy $N$ tubes and Incoloy shell 43
Steam generators
115
76
Steam reheaters 82
54
Reheat steam preheaters
149
none
in this study and in the reference design report is based on published costs of similar pumps (as adjusted for capacity and head requirements), on MSRE pump cost experience, and on the basis of considerable intuitive judgment. Table 11 indicates the pumping requirements which served as a basis for assuming allowances for the pump costs in the modified MSBR plant.

Use of the third circulating salt system would add four pumps of about 2700 hp each, would reduce the power requirements of another set of four pumps from 3200 hp to 1800 hp each, and would eliminate the need for the two 6000-hp each pressure-booster pumps in the feedwater system. As shown in Table 12, the connected load of the pump motors is reduced by a total of about $5,400 \mathrm{~kW}(\mathrm{e})$ in the modified system. If it is assumed that all the pumping energy is usefully converted to heat, about $5,400 \mathrm{~kW}(\mathrm{t})$ is thus not available in the modified system for conversion into electric power at the average overall plant efficiency of 44. $4 \%$. The net savings in auxiliary electric load is thus about 3,000 $\mathrm{kW}(\mathrm{e})$. With power worth $5.3 \mathrm{mills} / \mathrm{kWhr}$, and $80 \%$ plant factor, this amounts to about $\$ 111,000 /$ year. At $13.7 \%$ fixed charges, the savings is equivalent to a plant investment of about \$817,000. Credit for this has been taken in Parts $A$ and $B$ of Table 1.

Table ll. Estimated Design Data and Allowances for Installed Costs of Salt-Circulating Pumps

|  |  |  |  | Modified MSBR |  |
| :--- | :---: | :---: | :---: | :---: | :---: |
|  | Fuel-Salt <br> Pumps | Secondary- <br> Salt Pump <br> Ref. MSBR | Second.ary- <br> Salt Pump | Tertiary- <br> Salt Pump |  |
| For each of 4 pumps: |  |  |  |  |  |
| Actual capacity, gpm | 14,255 | 18,768 | 13,380 | 17,372 |  |
| Nominal capacity, gpm | 16,000 | 20,000 | 16,000 | 20,000 |  |
| Average salt density, lb/ft | 208 | 117 | 124 | 105 |  |
| Estimated total head, fta | 150 | 300 | 230 | 300 |  |
| Estimated horsepower | 2360 | 3210 | 1800 | 2680 |  |
| Cost allowance, in \$1000, | $\$ 3300$ | $\$ 4400$ | $\$ 2750$ | $\$ 3800$ |  |
| for total of 4 pumps |  |  |  |  |  |

${ }^{a_{\text {Estimate }}}$ based on calculated $\Delta p^{\prime}$ s in heat transfer equipment.
${ }^{6}$ Cost assumed to be in proportion to capacity and horsepower requirement.

Table 12. Estimated Pumping Power Requirements and Worth of Improved Efficiency of Modified MSBR Cycle

|  | Reference Design MSBR | Modified MSBR With Third Loops |
| :---: | :---: | :---: |
| Total pumping power, kW(e): |  |  |
| Pressure-booster pumps | 9,200 | none |
| Fuel-salt pumps | 7,039 | 7,039 |
| Secondary-salt pumps | 9,575 | 5,369 |
| Tertiary-salt pumps $\frac{n}{28}$ | $\frac{\text { none }}{28,814}$ | $\frac{7,994}{20,402}$ |
| Savings in pump power with modified system, $\mathrm{kW}(\mathrm{e})$ | ystem, | 5,400 |
| Difference in heat inputs to systems from pump work, $\mathrm{kW}(\mathrm{t})$ | from | 5,400 |
| Electric power potential of $5,400 \mathrm{~kW}(\mathrm{t})$ at $44.4 \%$ thermal efficiency, kW(e) | ) at $2,$ | 2,400 |
| Net savings in power with modified cycle, $\mathrm{kW}(\mathrm{e})$ | cle, $3,0$ | 3,000 |
| Capital cost worth of $3,000 \mathrm{~kW}(\mathrm{e})$ at $80 \%$ plant factor, $13.7 \%$ fixed charges, and power worth $5.3 \mathrm{mills} / \mathrm{kWhr}$ | $\begin{array}{ll} 80 \% & \$ 81 \\ \text { and } \end{array}$ | \$817,000 |

## 5. SALT INVENTORY COSTS

The modified primary heat exchangers will contain about $56 \mathrm{ft}^{3}$ more fuel salt than those used in the reference design, as indicated in Table 13. On the basis of the $\$ 57 /$ lb fuel-salt cost used in ORNL-4541, this amounts to an additional investment of $\$ 671,000$ for the MSBR plant. Following the procedures used in the reference report, however, this capital cost is not included in the plant capital cost but in the fuelcycle cost. This would increase the fuel-cycle cost by about 0.013 mills/kWhr. (In both the reference and the modified plant designs it was assumed that the cleanup costs for the fuel salt at the end of the 30 -year plant life would be great enough to make it have essentially no resale, or "scrap" value.)

The estimated price of the nitrate-nitrite salt used in the modified design is 15 cents/lb as compared to 50 cents/ 1 b for the sodium fluoroborate used in the reference design. Both of these salts are assumed to have no resale value at end of the useful life of the plant.

As shown in Table 14, the estimated volume of the $\mathrm{LiF}^{\mathrm{F}} \mathrm{BeF}_{3}$ used in the secondary system is about $3200 \mathrm{ft}^{3}$. Almost three-fourths of this is in the shell-side of the primary heat exchangers. Using the same prices as in ORNL-4541, where ${ }^{7} \mathrm{Li}$ is assumed to cost $\$ 120 / \mathrm{kg}$, and ${ }^{7} \mathrm{LiF}$ and $\mathrm{BeF}_{a}$ to cost $\$ 16.50$ and $\$ 7.50 / \mathrm{Ib}$, respectively, the estimated cost of ${ }^{7} \mathrm{LiF}-\mathrm{BeF}$ a is about $\$ 12 / 1 \mathrm{~b}$. The total estimated cost of the secondary salt inventory is about $\$ 4,800,000$, as shown in Table 13. It is assumed that the salt will last the lifetime of the plant without reprocessing or replacement costs. At the end of 30 years it is assumed that the salt will have a resale value of $50 \%$, or $\$ 6 / \mathrm{Ib}$. (The salt could be used as the secondary coolant in another MSBR or as the carrier to make up new batches of fuel salt.) The present worth of $\$ 2,400,000$ thirty years hence at $8 \%$ interest is $\$ 239,000$, and credit for this has been taken in Table 1.

Table 13. Estimated Salt Inventory Costs

|  | Reference Design MSBR | Modified MSBR With Third Loops |
| :---: | :---: | :---: |
| Fuel salt | ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{\mathbf{a}}-\mathrm{ThF}_{4}-\mathrm{UF}_{4}$ | ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{2}-\mathrm{ThF}_{4}-\mathrm{UF}_{4}$ |
| Total volume, ${ }^{\text {a }} \mathrm{ft}^{3}$ | 2200 | 2256 |
| Total weight, lb | 457,000 | 469,000 |
| Total cost ${ }^{\text {b }}$ | \$23,533,000 | \$24,204,000 |
| Resale value after 30 yr | 0 | 0 |
| Secondary salt | $\mathrm{NaF}-\mathrm{NaBF}_{4}$ | ${ }^{7} \mathrm{LiF}-\mathrm{BeF}_{3}$ |
| Total volume, $\mathrm{ft}^{3}$ | 8400 | $3200{ }^{\text {c }}$ |
| Total weight, lb | 1,000,000 | 397,000 |
| Average cost, $\$ / \mathrm{Ib}$ | \$0.50 | \$12 ${ }^{\text {d }}$ |
| Total cost | \$500,000 | \$4,800,000 |
| Resale value after 30 yr | 0 | \$2,400,000 |
| Present worth, at $8 \%$ |  | \$239,000 |
| Tertiary salt |  | $\mathrm{KNO}_{3}-\mathrm{NaNO}_{2}-\mathrm{NaNO}_{3}$ |
| Total volume, $\mathrm{ft}^{3}$ |  | $8400{ }^{\text {e }}$ |
| Total weight, 1 b | none | 900,000 |
| Average cost, \$/Ib |  | \$0.15 |
| Total cost |  | \$135,000 |
| Resale value after 30 yr |  | 0 |

${ }^{a}$ Includes $480 \mathrm{ft}^{3}$ in chemical processing plant.
$b_{\text {Based }}$ on fertile salt cost of about $\$ 57 / 1 \mathrm{~b}$ and an average inventory value of $\$ 31 / \mathrm{lb}$ in the chemical plant.
${ }^{c}$ See Table 11.

${ }^{e}$ Assumed to have same volume as reference design secondary system.

Table 14. Estimated Volume of $\mathrm{LiF}^{-\mathrm{BeF}_{2}}$ Salt in Secondary System of Modified MSBR

Primary heat exchanger volumes

| Shell, $\mathrm{ft}^{3}$ per unit | $+762 \mathrm{ft}^{3}$ |
| :--- | :--- |
| Head, $\mathrm{ft}^{3}$ per unit | +13 |
| Tubes, $\mathrm{ft}^{3}$ per unit | -123 |
| Liner, $\mathrm{ft}^{3}$ per unit | -95 |
| Downcomer | -5 |
| Baffles | -17 |
| $90^{\circ}$ outlet bends | +19 |
| Net volume one unit | $554 \mathrm{ft}^{3}$ |

Total volume 4 units
2,216 $\mathrm{ft}^{3}$
Secondary heat exchanger volumes (tube side)
Tubes $\quad 133 \mathrm{ft}^{3}$
Head allowance
20
Volume in one unit $153 \mathrm{ft}^{3}$
Total volume in 4 units 612
Secondary salt piping volumes
Volume of $35-\mathrm{ft} 20-\mathrm{in}$. pipe per unit, for total 4 units 306

Drain tank heel allowance
$\frac{66}{3,200} \mathrm{ft}^{3}$
Total estimated volume of salt

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[^0]:    ${ }^{1}$ Roy C. Robertson et al., Conceptual Design of a Single-Fluid MoltenSalt Breeder Reactor, $\overline{\text { RNIL-4541 (May 1971). }}$

