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SOLUTION OF THE EQUATION DESCRIBING THE INTERFACE BETWEEN TWO

FLUIDS FOR THE VOLUME AND PRESSURE WITHIN ATTACHED,

SESSILE SHAPED, BUBBLES AND DROPS

J. W. Cooke

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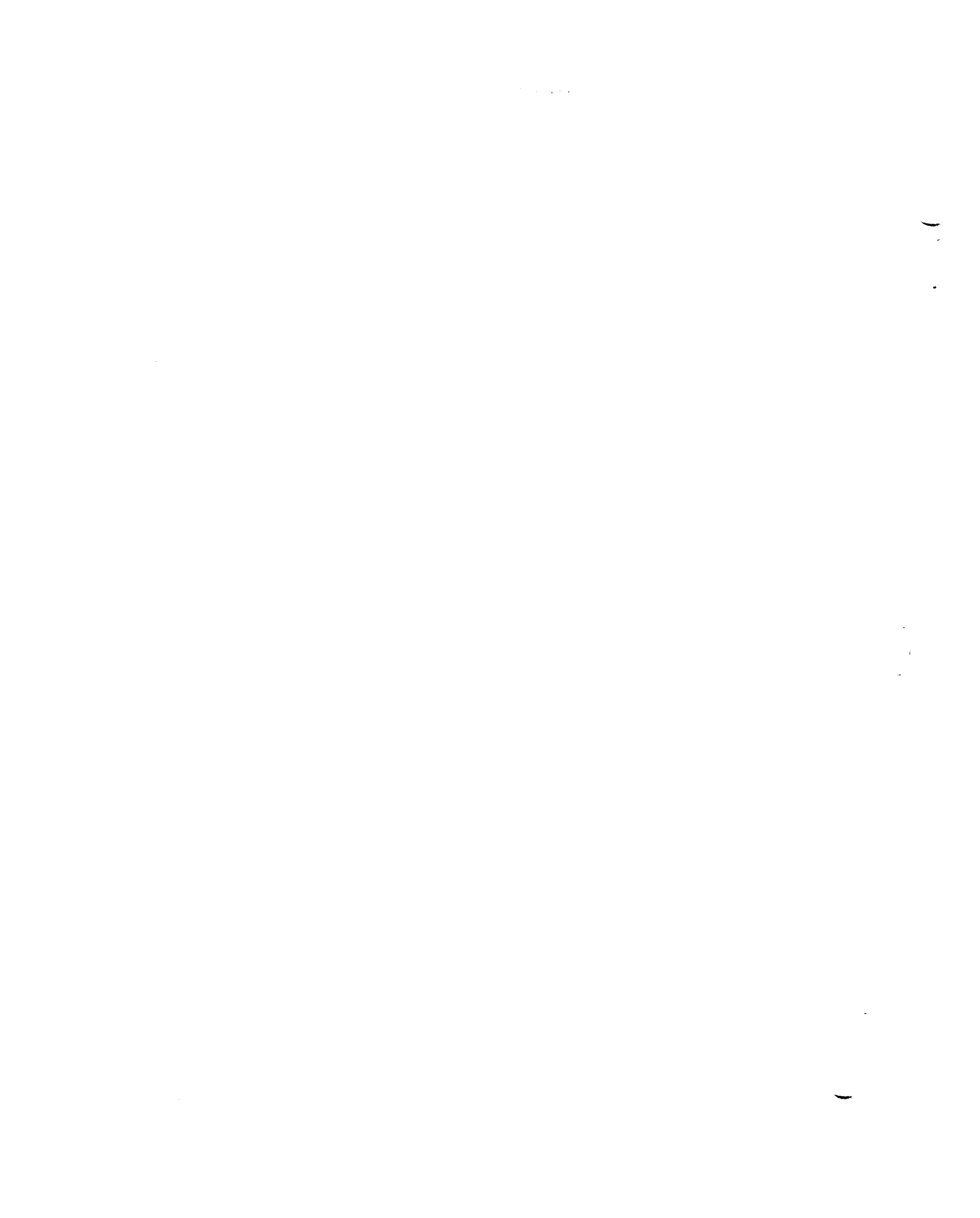U. S. ATOMIC ENERGY COMMISSION

CONTENTS

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ABSTRACT

A numerical computer program was written to solve the equation describing the interface between two immiscible fluids to obtain the shape, size, volume, and pressure of attached bubbles and droplets. These relationships are important to the study of three-phase heat transfer, superheat, critical constants, and interfacial energies. Previous solutions have been obtained with limited accuracy for a restricted number and range of variables. The present results are given in both graphical and tabular form for a wide range and number of parameters, and the computer program is included so that an even broader range and number of variables, as well as specific values, can be obtained as required. In particular, an expression for the maximum-bubble-pressure was derived which is considerably more accurate over a wider range than previous expressions.

Keywords. Surface tension, interfacial tension, bubbles, drops, contact angle, maximum-bubble-pressure, fluid-vapor interface.

INTRODUCTION

A knowledge of relationships among volume, pressure, shape, and size of attached bubbles and droplets are important in the study of boiling and condensing heat transfer, superheat and critical point phenomena, mass transfer studies, and in the measurement of contact angles and surface tension.[1-4] These relationships can be obtained from the solution of the second-order, nonlinear differential equation describing the interface between two fluids. This equation has been solved by perturbation analysis for very small bubble sizes by Rayleigh[5] and Schrödinger,[6] by numerical hand calculations for a wide range of discrete drop shapes by Bashforth and Adams,[7] and by numerical computer calculations for droplet volumes and heights by Baumeister and Hamill.[8] However, the Rayleigh-Schrödinger solution is no longer used extensively because of concern regarding its accuracy; and the Bashforth and Adams (as well as the

Baumeister and Hamill) solutions were obtained for a limited number and range of variables.

In order to check the accuracy and to extend the usefulness of the above solutions, a numerical computer program was written to solve the interfacial equation. The subject program solves the interfacial equation for any value of the dimensionless shape parameter, β, for positive values of the term βZ (sessile drops or above-attached bubbles)[*] and extends the previous solutions for $\phi > 180$ degrees. The solution not only provides the dimensionless X, Z, ϕ coordinates describing the profiles of the interface, but also gives the bubble volume within and the pressure difference across the interface as a function of profile shape and, in addition, the maximum pressure difference across the interface for a given radius of attachment (which is required for the calculation of the surface tension by the maximum-bubble-pressure technique).

This report describes the derivation of the interfacial equation, its numerical solution, and the computer program employed. The results and an estimation of their accuracy are presented and compared with the previous solutions.

DERIVATION OF THE INTERFACIAL EQUATION

At the interface separating two immiscible fluids (usually a liquid and its vapor), a force imbalance exists which results in an interfacial tension. The tension is a force per unit length pulling uniformly in all directions tangent to the interface. Neglecting gravitational forces, a force balance on an infinitesimal segment of a free surface is shown in Fig. 1-a. The surface is bowed by a uniform differential pressure, P, where r_1 and r_2 are the principal radii of curvature. The force balance would thus be:

$$P \, \Delta x \, \Delta z = 2\gamma \left(\Delta x \, \frac{\Delta z}{2r_1} + \Delta z \, \frac{\Delta x}{2r_2} \right) , \tag{1}$$

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[*]The case for negative values of the term βZ (pendant drops or below-attached bubbles) will be presented in a later report.
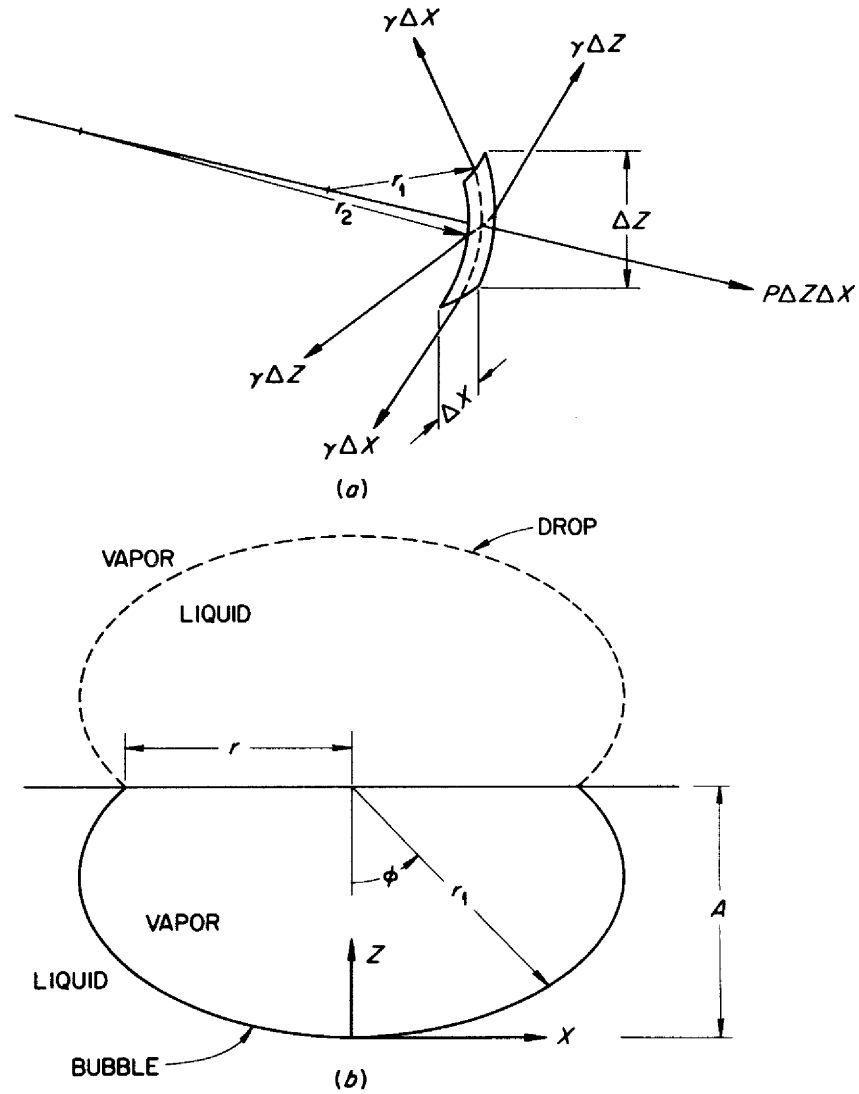
Fig. 1. Schematic View of a Sessile-Type Drop and Bubble and the Force Balance on an Infinitesimal Surface Element.

or

$$P = \gamma \left(\frac{1}{r_1} + \frac{1}{r_2} \right) \quad . \tag{2}$$

If gravitational forces are present, the influence of the differential fluid density, ρ, must be considered. For an interface assumed to be symmetrical about an axis of revolution, the force balance equation for the bubble shown in Fig. 1-b will be

$$P - \gamma \left(\frac{1}{r_1} + \frac{1}{r_2} \right) - \frac{\rho g}{g_c} (A - z) = 0 \quad , \tag{3}$$

where $\rho = \rho_L - \rho_V$. At the orgin, $r_1 = r_2 \equiv b$ and thus

$$P = \frac{2\gamma}{b} + A \frac{\rho g}{g_c} \quad . \tag{4}$$

Furthermore, with $r_1 = x/\sin \phi$, Eq. (3) becomes

$$\frac{2\gamma}{b} + \frac{\rho g z}{g_c} - \gamma \left(\frac{\sin \phi}{x} + \frac{1}{r_2} \right) = 0 \quad . \tag{5}$$

A similar derivation for the drop shown in Fig. 1-b would also result in Eq. (5). This equation can now be made dimensionless with respect to b, and after rearranging:

$$\frac{1}{R} + \frac{\sin \phi}{X} = 2 + \beta Z \quad , \tag{6}$$

where

$$\beta \equiv \frac{g \rho b^2}{\gamma g_c} \tag{7}$$

and

$$R = \frac{r_2}{b} \ , \quad X = \frac{x}{b} \ , \quad \text{and} \quad Z = \frac{z}{b} \quad .$$

Equation (6) can be transposed into Cartesian coordinates by substituting

$$\frac{1}{R} \equiv \frac{d^2Z/dX^2}{[1 + (dZ/dX)^2]^{3/2}}$$

and

$$\sin \phi \equiv \frac{dZ/dX}{[1 + (dZ/dX)^2]^{1/2}}$$

to obtain

$$\frac{d^2Z}{dX^2} + \left[1 + \left(\frac{dZ}{dX}\right)^2\right]\frac{dZ}{XdX} = (2 + \beta Z)\left[1 + \left(\frac{dZ}{dX}\right)^2\right]^{3/2} \quad , \tag{8}$$

which is a second-order nonlinear differential equation, with boundary conditions:

$$X = 0 \quad ; \quad Z = 0$$

$$X = 0 \quad ; \quad dZ/XdX = 1$$

Although Eq. (8) cannot be solved analytically in terms of ordinary functions, its numerical solution is described in the next section.

NUMERICAL SOLUTION OF THE INTERFACIAL EQUATION

Equation (6) can be rearranged to read

$$R = \frac{1}{2 + \beta Z - (\sin \phi)/X} \tag{9}$$

and by definition:

$$\frac{d\phi}{d\phi} \equiv 1 \quad ,$$

$$\frac{dX}{d\phi} \equiv R \cos \phi = F(X, Z, \phi) \quad , \tag{10}$$

$$\frac{dZ}{d\phi} \equiv R \sin \phi = G(X, Z, \phi) \quad . \tag{11}$$

As long as $\beta Z > 0$, R will always be finite.

A numerical technique of fourth-order accuracy developed by Runge-Kutta[9] was selected to solve the set of simultaneous Eqs. (9), (10), and (11); the iterative equations decribing this technique are:

$$X_{n+1} = X_n + \frac{1}{6}(k_0 + 2k_1 + 2k_2 + k_3) + 0(\Delta\phi)^5 \quad , \tag{12}$$

$$Z_{n+1} = Z_n + \frac{1}{6}(m_0 + 2m_1 + 2m_2 + m_3) + 0(\Delta\phi)^5 \quad , \tag{13}$$

$$\phi_{n+1} = \phi_n + \Delta\phi \quad ,$$

where

$$k_0 = \Delta\phi \, F(X_n, Z_n, \Delta\phi) \quad ,$$

$$k_1 = \Delta\phi \, F(X_n + \frac{1}{2}k_0, Z_n + \frac{1}{2}m_0, \phi + \frac{1}{2}\Delta\phi) \quad ,$$

$$k_2 = \Delta\phi \, F(X_n + \frac{1}{2}k_1, Z_n + \frac{1}{2}m_1, \phi + \frac{1}{2}\Delta\phi) \quad ,$$

$$k_3 = \Delta\phi \, F(X_n + k_2, Z_n + m_2, \phi + \Delta\phi) \quad ,$$

$$m_0 = \Delta\phi \, G(X_n, Z_n, \Delta\phi) \quad ,$$

$$m_1 = \Delta\phi \, G(X_n + \frac{1}{2}k_0, Z_n + \frac{1}{2}m_0, \phi + \frac{1}{2}\Delta\phi) \quad ,$$

$$m_2 = \Delta\phi \, G(X_n + \frac{1}{2}k_1, Z_n + \frac{1}{2}m_1, \phi + \frac{1}{2}\Delta\phi) \quad ,$$

$$m_3 = \Delta\phi \, G(X_n + k_2, Z_n + m_2, \phi + \Delta\phi) \quad ,$$

and where the symbol $0(\Delta\phi)^5$ represents a term which is small, of the order $(\Delta\phi)^5$, when $\Delta\phi$ is small.

Equations (12) and (13) are of a form that can be readily transformed into a computer program.

COMPUTER PROGRAM

The computer program for the solution of the interfacial equation and for the calculation of the various output parameters is listed in Appendix C. The program consisted of four parts which are discussed below.

Solution of the Differential Equation. Two subroutines in both single and double precision and consisting of four iteration loops were written to solve Eqs. (12) and (13). The subroutines RHOS and RHOD calculate the values of R and supply the values of the functions F and G to the subroutines RUNGKS and RUNGKD. These latter subroutines calculate the values of the coefficients k_i and m_i and the new values of X, Z, and ϕ for reintroduction in RHOS and RHOD to continue the iterative procedure. The iteration procedure is initiated by equating X, Z, and ϕ to zero and R to one.

Solution for h/a and V/a^3 versus β and ϕ. The pressure and volume within the attached bubble are calculated from X, Z, and ϕ. The pressure relation is given by Eq. (4) and can be simplified by using Eq. (7) and the definition of the specific cohesion,

$$a^2 \equiv \frac{2\gamma g_c}{\rho g} , \tag{14}$$

to obtain

$$h/a = \sqrt{2/\beta} + Z_r \sqrt{\beta/2} , \tag{15}$$

where

and

$$Z_r = Z(x = r) = A ,$$

$$h = g_c P/\rho g .$$

The volume relation can be obtained from the integration of

$$d\left(\frac{V}{b^3}\right) = \pi X^2 \, dZ , \tag{16}$$

where X and dZ can be obtained from Eq. (6). Integrating the resulting equation by parts gives the relationship:

$$\frac{V}{b^3} = \frac{\pi X^2}{\beta}\left[2 + \beta Z - \frac{2\sin\phi}{X}\right] . \tag{17}$$

Upon substituting Eqs. (6), (14), and (15) into Eq. (17), the final expression for the dimesionless volume is obtained:

$$\frac{V}{a^3} = \pi\frac{r}{a}\left[\left(\frac{r}{a}\right)\left(\frac{h}{a}\right) - \sin\phi\right] , \tag{18}$$

where

$$\frac{r}{a} = X_r\sqrt{\beta/2}$$

is the dimensionless radius of attachment.

Solution for h/a and V/a³ versus β and r/a. To obtain the pressure and volume within attached bubbles for a given radius of attachment from the numerical solutions X, Z, ϕ, of the interfacial equation, several conditional "IF" statements were required. The interrelationship of h/a, β, r/a, and ϕ are shown schematically in Fig. 2. The iterative solution of the interfacial equation proceeds along constant β lines for given steps of $\Delta\phi$. A conditional check is made to note the intersection of the given r/a curve (denoted by □), and the values at the point of crossing are determined by linear interpolation. Since r/a is multi-valued, an additional check is necessary to note when the increasing values of r/a start to decrease at $\phi = n\pi/2$, n = 1, 5 ... (denoted by ○), or when decreasing values start at $\phi = n\pi/2$, n = 3, 7 .... In this manner, solutions for many values of r/a can be obtained for a given value of β.

Solutions for h̄/a and V̄/a³ versus r/a. These solutions were somewhat more difficult to obtain than those described above since it was necessary to increment β as well as ϕ. Both of the conditional checks described above for crossing a given value of r/a and a change from increasing to decreasing r/a values were required as well as a check for the change from increasing to decreasing values of h/a (denoted by $\triangle$ in Fig. 2).
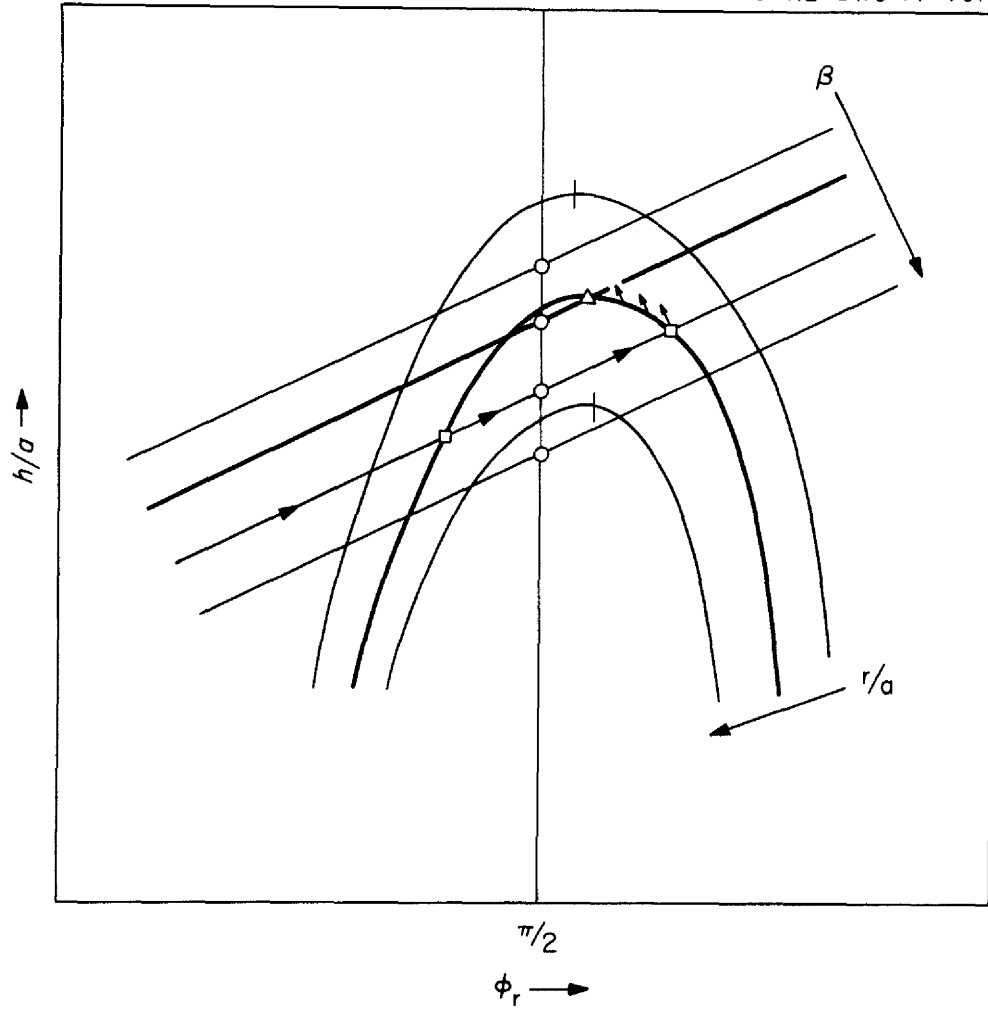
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Fig. 2. Schematic Representation of the Relationships Between h/a, β, r/a, and ϕ_r.

For a given value of β, h/a is a multi-valued function so that the choice of increasing or decreasing β to approach $\bar{h}/a$ must be carefully considered. Furthermore, the direction of approach to $\bar{h}/a$ along the given value of r/a must also be carefully considered. To insure the most trouble-free solution over the entire h/a versus φ field, a decremental approach from right to left (as shown by the arrows in Fig. 2) was chosen.

To reduce the number of iterations required to locate $\bar{h}/a$, an estimate of the value of β is calculated from equations fitted to a few preliminary results. The initial β value is then decremented along first a coarse grid, and finally along an extra fine grid to obtain the final solution using double precision.

Range, Running Time, and Accuracy

The ranges of the computer program as presently written are $0 < \phi < 360°$, $0.1 \le r/a \le 2.0$, and $0.02 \le \beta \le 150$; however, these can be easily extended at some sacrifice of either the running time or accuracy. The average running time for the program to obtain a value of $\bar{h}/a$ for a given value of r/a is approximately 10 seconds on the IBM 360-91 Computer system. The average running time for the other programs is considerably shorter per solution.

An estimate of the accuracy (to be given later) was obtained by comparing the values of h/a as a function of φ for various values of β, Δφ, and for single and double precision.

The results of the computer solution are discussed in the next section.

RESULTS

The results in both tabular and graphical form are presented in this section and in the Appendix.

The profile of a bubble for β = 0.8 is shown in Fig. 3. The profile is shown extended to φ = 360°, which would be possible if a suitable
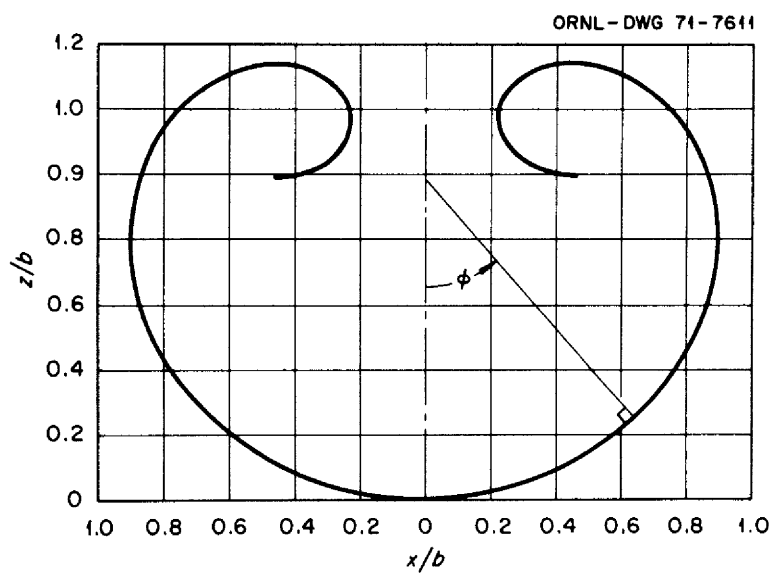
Fig. 3. Profile of a Bubble with β = 0.8 for 0° ≤ φ ≤ 360°.

form of attachment were provided, and the center of bouyancy were to remain on the z axis.

A tabulation of β, x/b, z/b, h/a, and V/a^3 for various values of ϕ are presented in Tables B.1 through B.6 (in Appendix B). Plots of h/a and V/a^3 versus ϕ for various r/a are shown in Figs. 4 and 5, respectively. Figure 4 clearly shows the attainment of a maximum value of h/a ($\overline{h}/a$) for a given radius of attachment. (This is the basis of the maximum-bubble-pressure technique[4] for the measurement of surface tension.) In addition, there is a minimum value of h/a as well as a maximum value for V/a^3.

The values of $\overline{h}/a$ and the corresponding values of $\overline{V}/a^3$, $\overline{\beta}$, $\overline{\phi}$, $\overline{x}/b$, $\overline{z}/b$ for various values of r/a are given in Table 1, and various cross-plots of these variables are given in Figs. A.1 through A.4 (Appendix A). These plots show that both $\overline{h}/a$ and $\overline{\phi}_r$ approach asymptotic values of $\sqrt{2}$ and $180°$,[*] respectively. Thus, the maximum pressure difference ($\overline{h}/a$) that a large tube can sustain will be very nearly independent of tube diameter.

A least-squares, polynomial fit of the computer solutions for various formulations of $\overline{h}/a$ and r/a were made. The forumlation that gave the best fit was

$$\frac{a}{\overline{h}} \frac{a}{r} = f\left(\frac{a}{\overline{h}} \frac{r}{a}\right), \tag{19}$$

which is of the same form as the perturbation solutions of Rayleigh-Schrödinger. For this reason, Eq. (19) was fitted to the data of Table 1 by the relationship:

$$F(y) = \frac{f(y) - i_0 + i_1 y + i_2 y^2}{y^3} = i_3 + i_4 y + i_5 y^2 + \cdots , \tag{20}$$

where i_0, i_1, and i_2 are the coefficients of the Rayleigh-Schrödinger solution and i_3, i_4, i_5, ... were determined by the least-squares procedure and are listed in Table 2.

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[*]These values can be obtained by the solution of Eq. (3) with $1/r_1 = 0$ and $1/b = 0$.
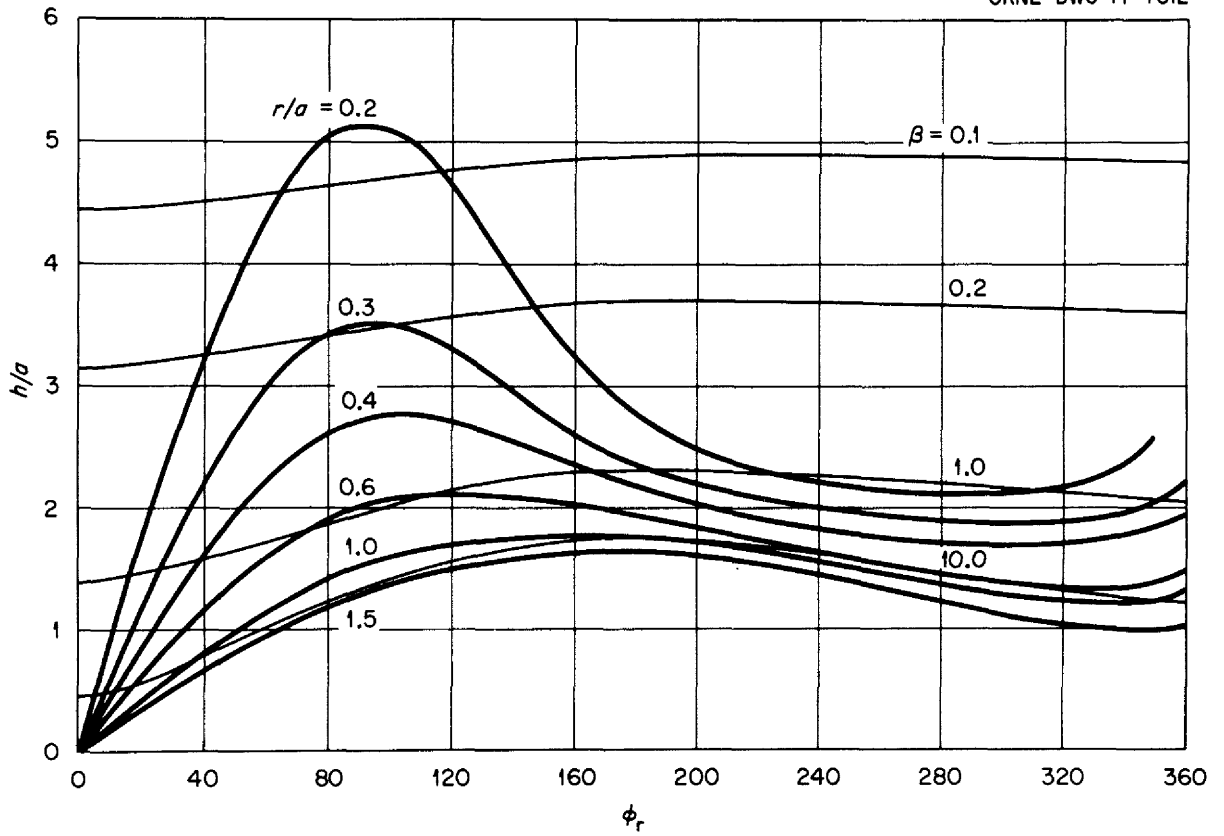
Fig. 4. Variation of the Dimensionless Pressure Difference, h/a, as a Function of ϕ_r for Various Values of β and the Dimensionless Radius of Attachment, r/a.
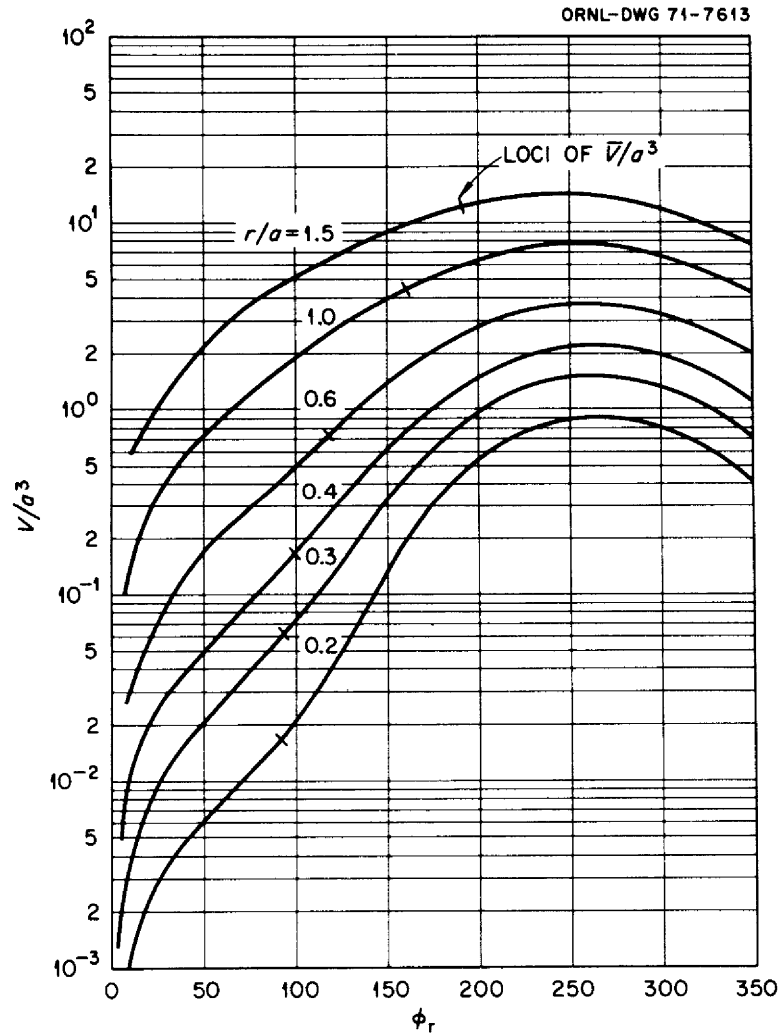
Fig. 5. Variation of the Dimensionless Volume, V/a^3, as a Function of ϕ_r for Various Values of the Dimensionless Radius of Attachment, r/a.

Table 1. Maximum Values of the Pressure ($\bar{h}/a$) and Corresponding
Values of the Size, Shape, and Volume of Bubbles as a
Function of the Radius of Attachment (r/a)

r/a	$\bar{\phi}$	$\bar{\beta}$	$\bar{x}/b$	$\bar{z}/b$	$\bar{h}/a$	$\bar{V}/a^3$
0.12	90.646	0.0290810	0.995156	0.999705	8.413519	0.003651
0.14	90.795	0.0397216	0.993412	0.998120	7.236465	0.005806
0.16	91.427	0.0521086	0.991243	1.00397	6.357323	0.008787
0.18	91.906	0.0662774	0.988791	1.00649	5.676507	0.012624
0.20	92.411	0.082286	0.986012	1.00865	5.134652	0.017439
0.22	92.906	0.100192	0.982929	1.00988	4.693892	0.023404
0.24	93.420	0.120066	0.979526	1.01064	4.328985	0.030656
0.26	93.945	0.141985	0.975814	1.01077	4.022445	0.039323
0.28	94.488	0.166037	0.971786	1.01039	3.761790	0.049570
0.30	95.474	0.192565	0.966824	1.01574	3.537926	0.062105
0.325	96.472	0.229022	0.960416	1.01752	3.299450	0.080293
0.350	97.476	0.269500	0.953463	1.01785	3.097815	0.101911
0.375	98.987	0.315035	0.944859	1.02327	2.925744	0.128876
0.400	100.225	0.365084	0.936222	1.02319	2.777713	0.158475
0.425	101.985	0.421688	0.925568	1.02746	2.649595	0.197373
0.450	103.483	0.483756	0.914990	1.02640	2.538105	0.239818
0.475	104.821	0.551689	0.904402	1.02173	2.440625	0.286173
0.50	107.164	0.631617	0.889730	1.02464	2.355273	0.346254
0.55	111.986	0.822982	0.857398	1.02159	2.214232	0.501889
0.60	117.488	1.072133	0.819487	1.00938	2.104842	0.708170
0.65	123.990	1.407182	0.774913	0.986809	2.019914	0.987762
0.70	130.993	1.853431	0.727151	0.950582	1.953875	1.347727
0.75	137.489	2.414075	0.682654	0.902832	1.902104	1.768822
0.80	140.664	2.977900	0.655617	0.853059	1.860445	2.136163
0.90	152.491	4.841565	0.578448	0.742939	1.798649	3.270626
1.00	158.492	7.100181	0.530738	0.648972	1.753511	4.356567
1.10	162.989	10.070060	0.490221	0.567348	1.718720	5.521721
1.20	165.994	13.866110	0.455742	0.497922	1.690847	6.736407
1.30	167.994	18.715220	0.424972	0.438376	1.667904	8.005399
1.40	169.991	25.16100	0.394711	0.385323	1.648640	9.386485
1.50	171.489	33.48783	0.366575	0.339160	1.632205	10.839000
1.60	172.494	44.16429	0.340486	0.299033	1.618007	12.356187
1.70	173.488	58.14618	0.315285	0.263383	1.605608	13.971941
1.80	173.998	75.94175	0.292110	0.232455	1.594682	15.640580
1.90	174.987	99.7358	0.268978	0.204340	1.584975	17.453834
2.00	175.489	130.1581	0.247919	0.180031	1.576295	19.314144

Table 2. Coefficients for the Polynomial Equations
Fitted to the Computer Solution

Coefficients	Eq. (19)	Eq. (21)
i_0	1.00000	-0.00090
i_1	-0.66667	1.04439
i_2	-0.66667	-0.47175
i_3	0.03230	1.43283
i_4	-5.52833	-4.59801
i_5	61.19134	5.38228
i_6	-351.38141	-2.73720
i_7	1099.76625	0.51837
i_8	-1930.93994	
i_9	1913.36384	
i_{10}	-1003.22519	
i_{11}	216.93848	

The main disadvantage of Eq. (19) is that an iterative procedure is necessary to calculate $a/\bar{h}$. A simpler formulation, but less accurate, was also fitted to the computer results:

$$\frac{a}{\bar{h}} = f\left(\frac{r}{a}\right). \qquad (21)$$

The coefficients for the polynomial fit of Eq. (21) are also listed in Table 2.

The accuracy of these two polynomial fits is discussed in the next section.

DISCUSSION OF RESULTS

An estimate of the accuracy of the computer results and comparisons with previous results are presented in this section.

Estimation of the Accuracy. Results were obtained for h/a versus ϕ for $\beta = 0.1$ and 100.0 with three values of $\Delta\phi = 1°$, $1/2°$, and $1/4°$ using both single and double precision. There was no change in h/a (to the seventh decimal place) as $\Delta\phi$ was decreased from $1°$ to $1/4°$ when double precision was used for $\beta = 0.1$ and only 0.00008% change for $\beta = 100.0$. The single precision results are plotted in Fig. 6, where the percent difference is with reference to the double precision, $\Delta\phi = 1/4°$, values. As can be seen, an interval less than $\Delta\phi = 1°$ decreased the accuracy of the results because of rounding errors when single precision was used.

Except for the maximum pressure results listed in Table 1, all the tabulated results were computed using single precision and interval $\Delta\phi = 1°$. The values listed in Table 1 were calculated using double precision and are good to at least the sevnth decimal place. The other tabulated results are good to at least the fifth decimal place.

Comparison with Previous Results. As anticipated, the careful, tedious hand calculations of Bashforth and Adams (which required a number of years to complete) were in good agreement (to the fifth decimal place) with the present computer solution. Trouble, however, develops when the Bashforth and Adams tables are singly and doubly interpolated to apply
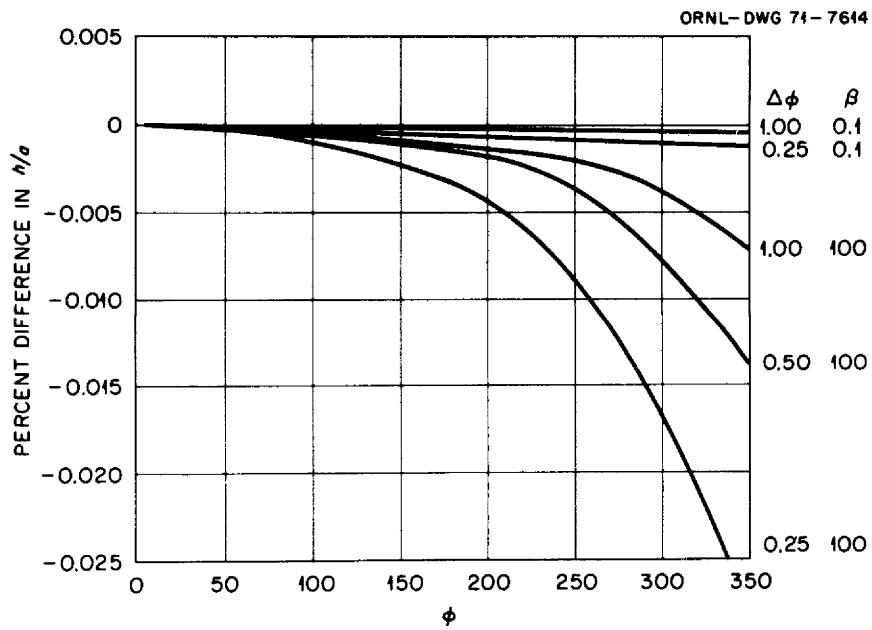
Fig. 6. Comparison of the Single Precision Results for h/a with the Double Precision, $\Delta\phi = 1/4$, Values as a Function of ϕ, $\Delta\phi$, and β.

their results to practical calculations. Sudgen,[10] "by careful inter-
polation" of Bashforth's tables, constructed a table for calculating
surface tension by the maximum-bubble-pressure method. (This table is
the one most often referred to in current literature on surface tension.)
The percent difference between our solution and Sudgen's as a function of
r/a is shown in Fig. 7. The maximum difference is -0.1%.

Although an error of 0.1% can be neglected for some studies at
elevated temperatures (where other errors are more significant), this
magnitude of error can be significant for many measurements made at room
temperature, where theoretical studies of small changes in the molecular
structure of the interface are being conducted. Futhermore, this error
can be magnified by as much as 20 times when the two tube, differential
technique is used to measure surface tension.

To be particularly noted in Fig. 7 is that the Rayleigh-Schrödinger
solution is in better agreement with the computer solution than Sudgen's
results all the way from $0 \leq r/a \leq 0.45$. This range of r/a covers a
large portion of the surface tension studies that have been conducted in
the past. In fact, the Rayleigh-Schrödinger equation is in error by less
than 1.05% all the way to r/a = 1.0. Thus, this much simpler analytic
solution can be used in many cases where precise surface tension values
are not needed.

Also shown in Fig. 7 are the deviations of Eqs. (19) and (21) from
the computer solutions. Equation (19) agrees to within ±0.05% all the
way to r/a = 1.5. The simpler Eq. (21) agrees to within ±0.07% from
$0.2 \leq r/a \leq 1.5$; but Schrödinger's equation is recommended for r/a < 0.2.
The main disadvantages of Eqs. (19) and (21) is that double precision
should be used in their solution, especially at the larger values of r/a.

Baumeister and Hamill presented their results as plots of droplet
volumes and heights as functions of droplet radii and contact angles.
In this form, their results could not be conveniently compared with the
present results. In addition, their results were given to only the
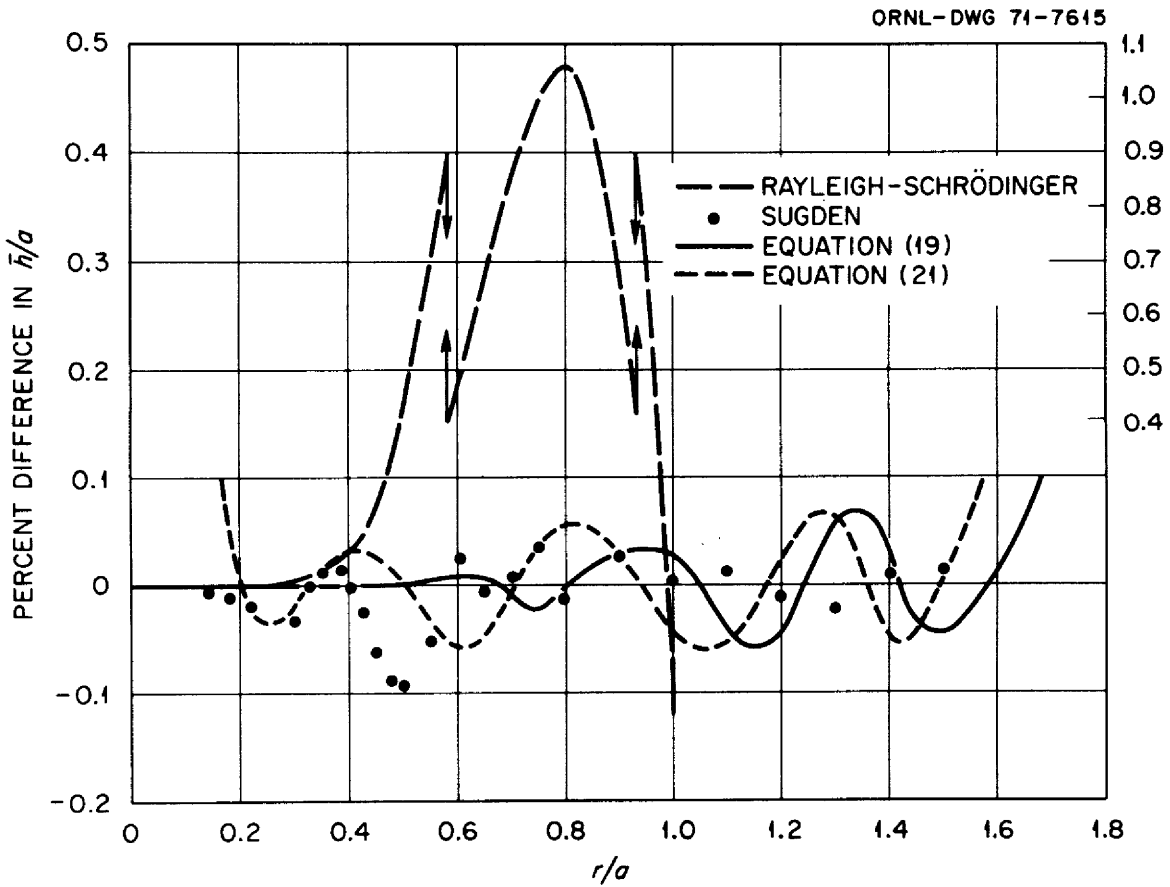third significant figure.

ORNL-DWG 71-7615

Fig. 7. Deviations of Previous Solutions and Present Polynomial Equations from the Present Computer Solution of the Maximum-Bubble Pressure.

CONCLUSIONS

A computer program was written to solve the second-order, nonlinear, differential equation describing the axially symmetric interface between two immiscible fluids. The present solution is limited to sessile-shaped drops and bubbles; the solution for pendant-shaped drops and bubbles requires a different approach and will be given in a later report. All of the present results are accurate to at least the fifth decimal place with some results accurate to the seventh decimal place.

The results are presented in a form that is useful in the analysis of boiling and condensing heat transfer, superheat, critical constants, and in the measurements of contact angles and surface tension. In addition, these results may be of use in the design of equi-stressed shells for containment vessels (i.e., above-ground water tanks, under-water storage of liquids and gases, lighter-than-air ballons, and submerged marine laboratories). The computer program is listed so that a wider range and number of variables can be obtained as desired.

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NOMENCLATURE

A	distance from the origin to the plane of attachment, cm
a^2	specific cohesion $\equiv 2\gamma g_c/\rho g$, cm^2
b	radius of curvature at the vertex of the bubble, cm
g	local acceleration due to gravity, cm/sec^2
g_c	dimensional constant, $dyne.sec^2/g.cm$
h	pressure differential across interface, cm of fluid with density ρ
$\bar{h}$	maximum value of h for given value of r/a, cm of fluid with density ρ
i	coefficients of polynomial equations
k	coefficients of numerical equations
m	coefficients of numerical equations
P	pressure differential across interface, $dyne/cm^2$
r	radius of circle of attachment, cm
r/a	dimensionless radius of attachment
r_1, r_2	principal radii of curvature of the interface, cm
R	dimensionless value of r_2, r_2/b
V	volume enclosed by the interface, cm^3

$\overline{V}$ volume enclosed by the interface for a given value of r/a at $h/a = \overline{h}/a$, cm^3

x   horizontal coordinate, cm

$\overline{x}$ value of x for a given value of r/a at $h/a = \overline{h}/a$, cm

X   dimensionless x, x/b

X_r value of X at x = r

y   independent variable in polynomial equation

z   vertical coordinate, cm

$\overline{z}$ value of z for a given value of r/a at $h/a = \overline{h}/a$, cm

Z   dimensionless z, z/b

Z_r value of Z at x = r

Greek Letters

β dimensionless parameter $\equiv g\rho b^3/g_c\gamma = 2(b/a)^2$

$\overline{\beta}$ value of β for a given r/a at $h/a = \overline{h}/a$

γ interfacial tension, dyne/cm

ϕ angle between axis of revolution and the normal to the surface

ϕ_r value of ϕ at the radius of attachment

$\overline{\phi}_r$ value of ϕ_r for a given r/a at $h/a = \overline{h}/a$

ρ positive density difference between the two fluids, g/cm^3
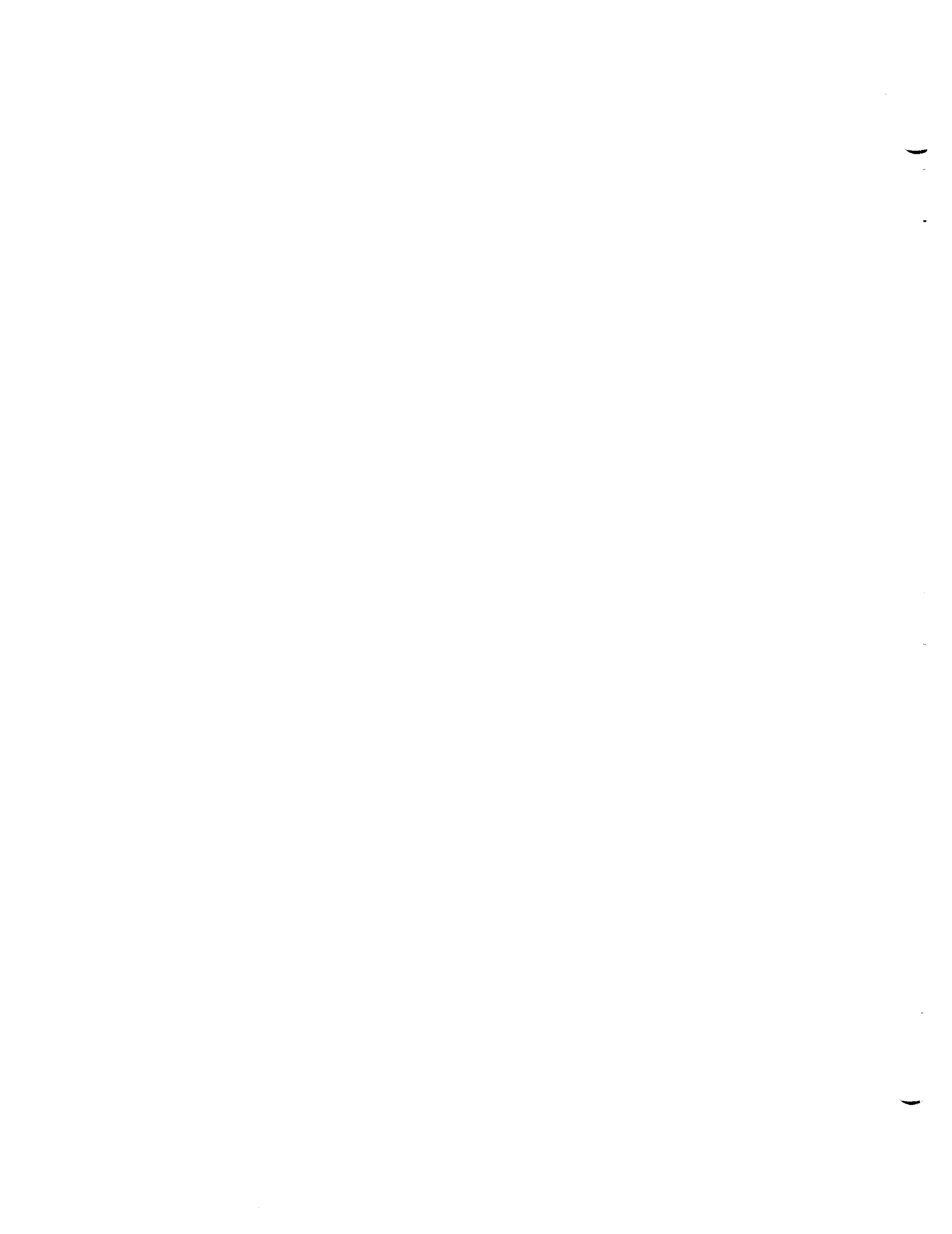
APPENDICES
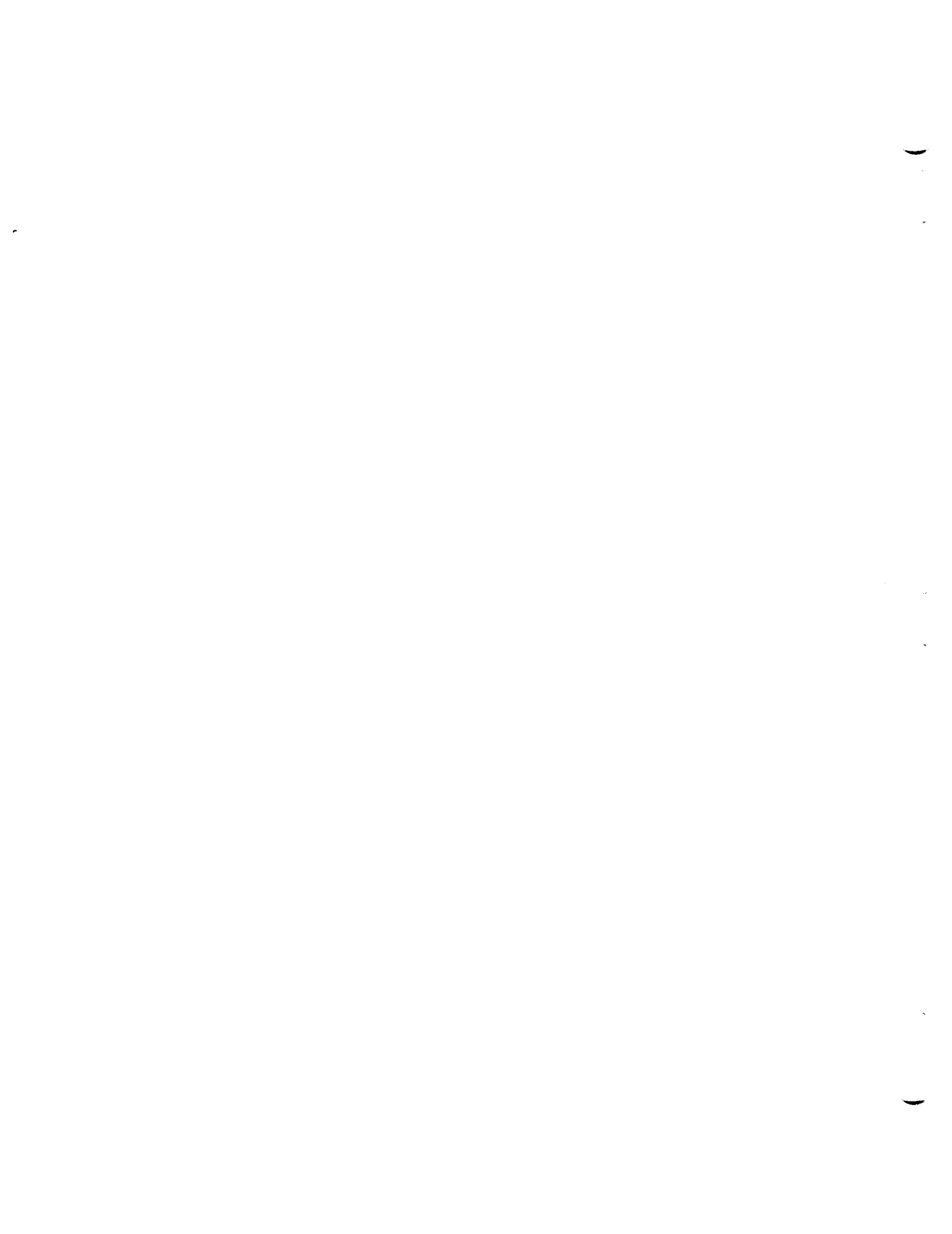
A. PARAMETRIC CROSSPLOTS

Various crossplots of the parameters shown in Figs. 4 and 5 are given in Figs. A.1, A.2, and A.3. The maximum bubble radius as a function of the angle ϕ_r for various radii of attachment is shown plotted in Fig. A.4. For $\phi_r < 90°$, the maximum bubble radius would be the radius of attachment. Also shown in Fig. A.4 is the loci of the maximum bubble radius where the maximum bubble pressure is reached.

ORNL-DWG 71-7616
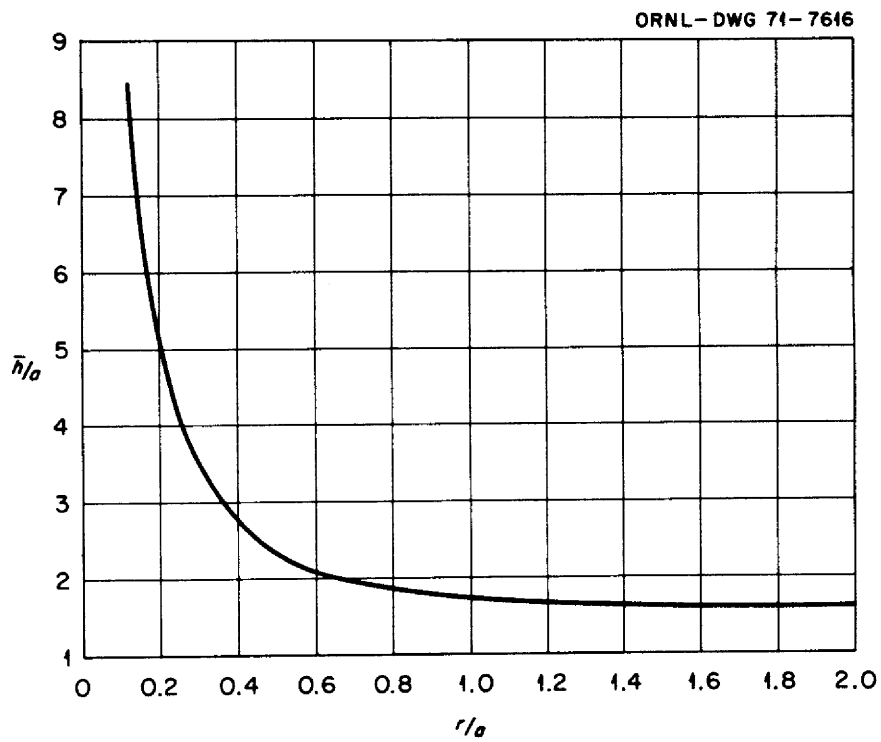
Fig. A.1. Dimensionless Maximum-Pressure-Difference Across the Interface Separating Two Fluids as a Function of the Dimensionless Radius of Attachment.
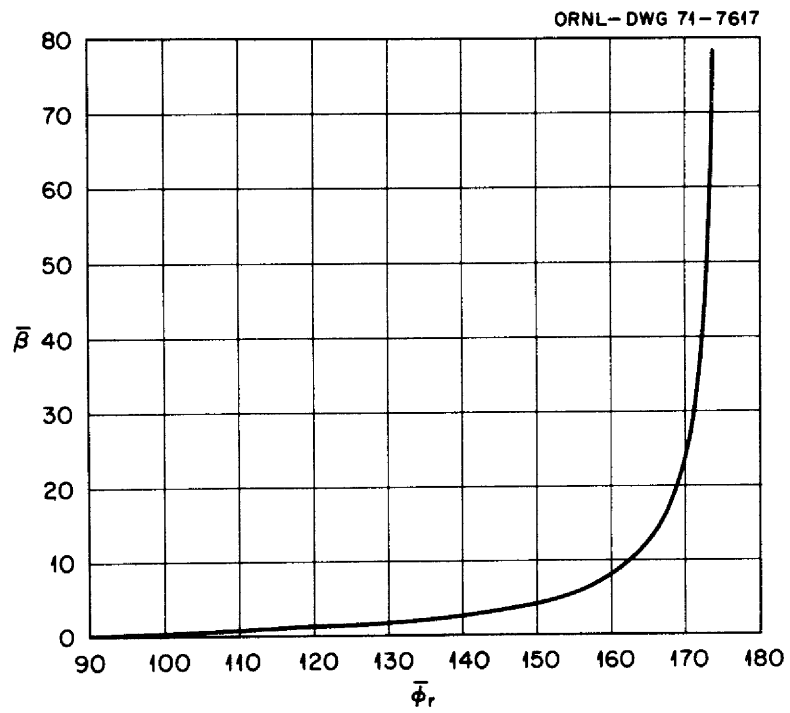
Fig. A.2. The Value of the Parameter β at $h/a = \bar{h}/a$ as a Function of $\bar{\phi}_r$.
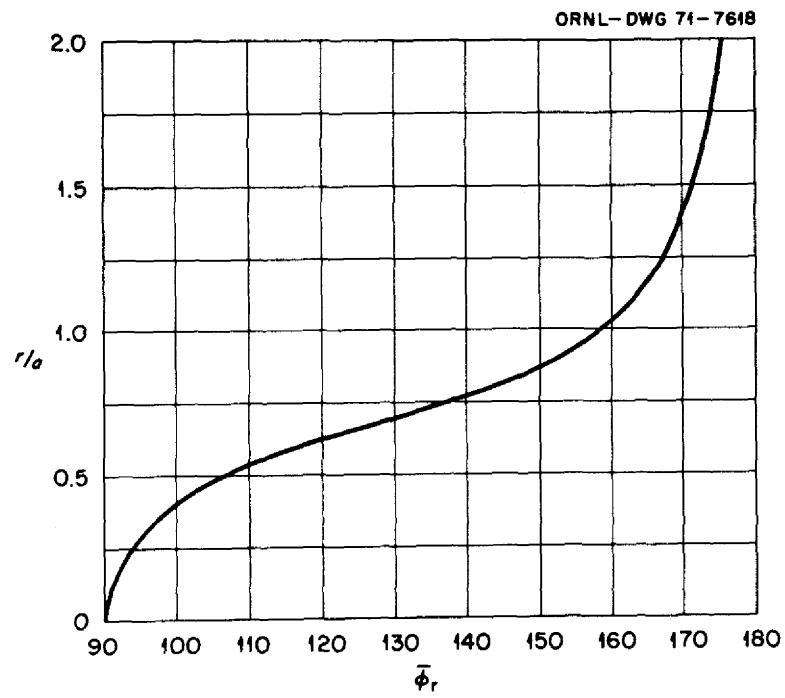
Fig. A.3. The Dimensionless Radius of Attachment as a Function of ϕ_r at $h/a = \bar{h}/a$.
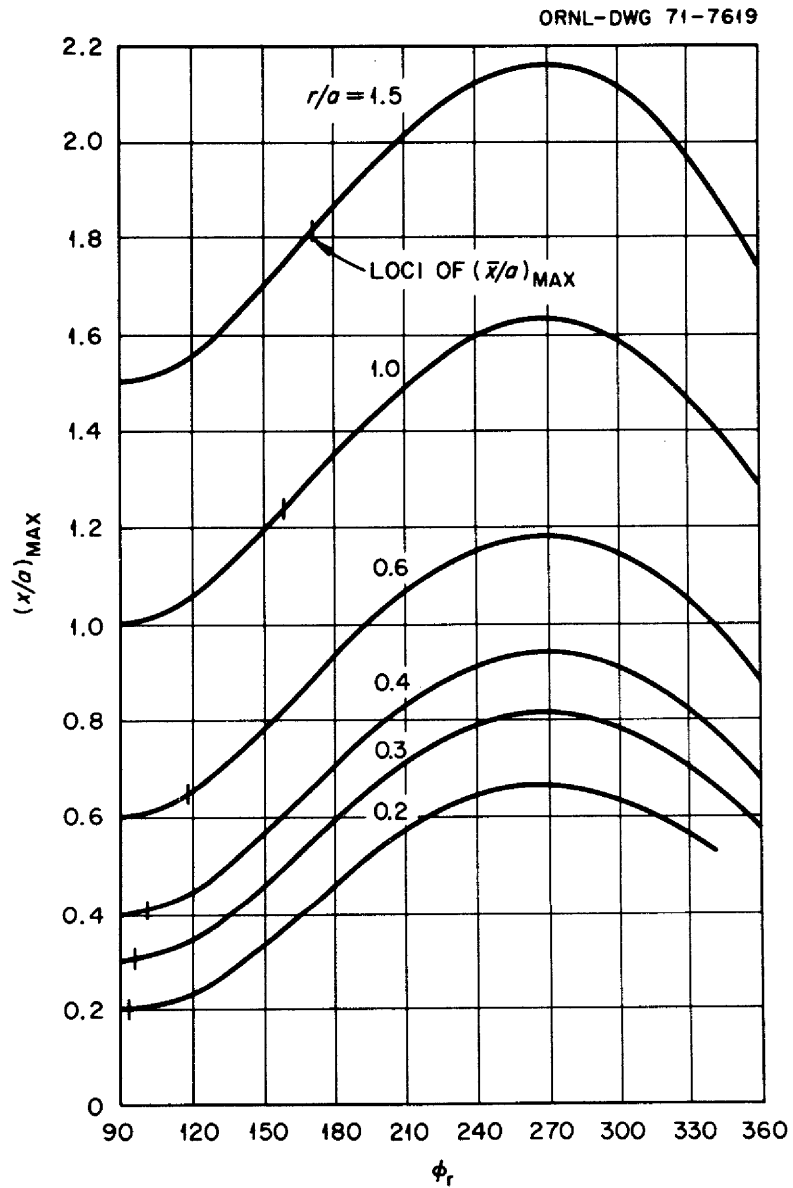
Fig. A.4. Maximum Radius of a Bubble (x/a at ϕ = 90°) as a Function of ϕ_r for Various Radii of Attachment (r/a). For ϕ_r < 90°, $(x/a)_{max}$ = r/a.

B.   TABULATED RESULTS

The computer results for the size, shape, pressure, and volume
of attached bubbles and drops for given radii of attachment are listed
in Table B.1 through B.6. The Results are arranged in ascending values
of ϕ_r.

Table B.1.　Size, Shape, Pressure, and Volume of Bubbles and
Drops with a Given Radius of Attachment

$$r/a = 0.2$$

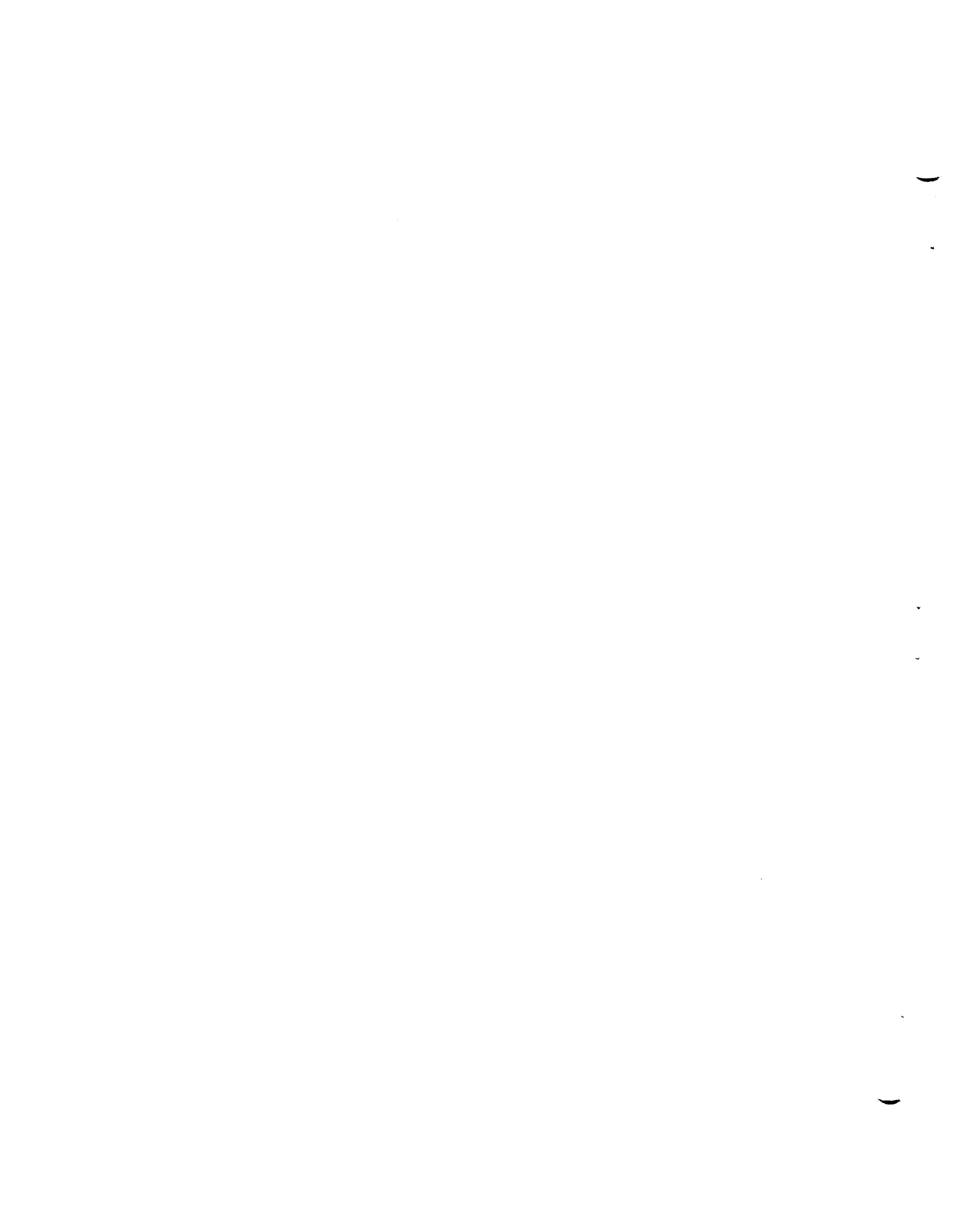| β	ϕ_r	x/b	z/b	h/a	V/a^3
30.00	2.99	0.05168	0.00135	0.26341	0.00030
18.00	3.86	0.06667	0.00225	0.34009	0.00042
8.00	5.80	0.10000	0.00505	0.51009	0.00063
5.50	7.00	0.12061	0.00734	0.61519	0.00077
4.00	8.21	0.14142	0.01013	0.72143	0.00090
3.50	8.78	0.15119	0.01158	0.77125	0.00096
2.90	9.66	0.16609	0.01400	0.84731	0.00106
2.30	10.86	0.18650	0.01765	0.95144	0.00119
2.00	11.66	0.20000	0.02317	1.02032	0.00128
1.80	12.30	0.21082	0.02262	1.07556	0.00135
1.50	13.49	0.23094	0.02717	1.17823	0.00149
1.10	15.81	0.26968	0.03727	1.37604	0.00175
0.95	17.05	0.29019	0.04327	1.48078	0.00189
0.80	18.63	0.31623	0.05161	1.61378	0.00207
0.66	20.59	0.34816	0.06295	1.77694	0.00229
0.58	22.04	0.37139	0.07194	1.89569	0.00247
0.50	23.84	0.40000	0.08399	2.04200	0.00268
0.46	24.92	0.41703	0.09166	2.12910	0.00281
0.42	26.17	0.43644	0.10089	2.22841	0.00296
0.38	27.62	0.45883	0.11220	2.34306	0.00313
0.34	29.35	0.48507	0.12636	2.47745	0.00334
0.30	31.46	0.51640	0.14463	2.63801	0.00361
0.25	34.87	0.56569	0.17662	2.89087	0.00405
0.23	36.59	0.58977	0.19386	3.01458	0.00427
0.21	38.60	0.61721	0.21486	3.15569	0.00455
0.19	40.99	0.64889	0.24102	3.31872	0.00489
0.17	43.91	0.68599	0.27474	3.51007	0.00531
0.15	47.60	0.73030	0.31993	3.73910	0.00585
0.14	49.86	0.75593	0.34889	3.87195	0.00622
0.13	52.51	0.78446	0.38410	4.02025	0.00664
0.12	55.69	0.81650	0.42796	4.18731	0.00719
0.11	59.65	0.85280	0.48479	4.37771	0.00792
0.10	64.87	0.89443	0.56306	4.59804	0.00898
0.09	72.70	0.94281	0.68614	4.85960	0.01079
0.0828	84.85	0.98295	0.88466	5.09473	0.01444

Table B.1 (Continued)

β	φ	x/b	z/b	h/a	V/a³
0.0828	95.17	0.98295	1.05331	5.12905	0.01878
0.09	107.58	0.94281	1.24471	4.97809	0.02660
0.10	115.80	0.89443	1.35781	4.77575	0.03445
0.11	121.43	0.85280	1.42604	4.59845	0.04173
0.12	125.82	0.81650	1.47280	4.44324	0.04890
0.13	129.46	0.78446	1.50654	4.30642	0.05606
0.14	132.59	0.75593	1.53162	4.18487	0.06328
0.15	135.34	0.73030	1.55053	4.07611	0.07058
0.17	140.07	0.68599	1.57565	3.88935	0.08542
0.19	144.07	0.64889	1.58951	3.73435	0.10060
0.21	147.60	0.61721	1.59609	3.60326	0.11610
0.23	150.78	0.58977	1.59777	3.49067	0.13189
0.25	153.70	0.56569	1.59596	3.39269	0.14794
0.34	164.94	0.48507	1.56461	3.07046	0.22259
0.38	169.34	0.45883	1.54451	2.96739	0.25668
0.42	173.53	0.43644	1.52274	2.87998	0.29109
0.46	177.57	0.41703	1.50000	2.80452	0.32574
0.58	189.15	0.37139	1.42956	2.62679	0.43005
0.66	196.74	0.34816	1.38256	2.53500	0.49949
0.70	200.55	0.33806	1.35938	2.49453	0.53405
0.80	210.35	0.31623	1.30143	2.40423	0.61955
0.90	220.90	0.29814	1.24333	2.32476	0.70350
1.00	233.12	0.28284	1.18282	2.25059	0.78543
1.10	250.16	0.26968	1.11197	2.17306	0.86409
1.10	289.74	0.26968	1.01358	2.10009	0.85531
1.00	306.55	0.28284	1.00345	2.12376	0.77164
0.90	318.54	0.29814	1.01038	2.16850	0.68854
0.80	328.88	0.31623	1.02769	2.23111	0.60513

Table B.2. Size, Shape, Pressure, and Volume of Bubbles and
Drops with Given Radius of Attachment

$r/a = 0.3$

β	ϕ_r	x/b	z/b	h/a	V/a^3
30.0	4.55	0.07746	0.00307	0.27011	0.00165
18.0	5.87	0.10000	0.00509	0.34859	0.00215
12.0	7.20	0.12247	0.00764	0.42700	0.00264
8.0	8.83	0.15000	0.01145	0.52291	0.00325
5.5	10.66	0.18091	0.01672	0.63076	0.00393
4.0	12.53	0.21213	0.02303	0.73967	0.00464
3.5	13.41	0.22678	0.02640	0.79085	0.00496
2.9	14.77	0.24914	0.03194	0.86891	0.00547
2.3	16.63	0.27975	0.04045	0.97588	0.00618
2.0	17.87	0.30000	0.04663	1.04663	0.00667
1.8	18.88	0.31623	0.05197	1.10340	0.00745
1.7	19.45	0.32540	0.05512	1.13547	0.00727
1.5	20.76	0.34641	0.06271	1.20901	0.00779
1.25	22.85	0.37947	0.07579	1.32483	0.00861
1.2	23.35	0.38730	0.07908	1.35225	0.00881
1.1	24.46	0.40452	0.08662	1.41264	0.00925
1.0	25.74	0.42426	0.09574	1.48191	0.00977
0.95	26.46	0.43529	0.10110	1.52063	0.01005
0.9	27.24	0.44721	0.10704	1.56252	0.01038
0.8	29.05	0.47434	0.12135	1.65789	0.01113
0.7	31.28	0.50709	0.14017	1.77323	0.01206
0.66	32.32	0.52223	0.14943	1.82662	0.01250
0.58	34.79	0.55709	0.17222	1.94970	0.01358
0.5	37.92	0.60000	0.20334	2.10167	0.01499
0.46	39.86	0.62554	0.22365	2.19240	0.01588
0.42	42.13	0.65465	0.24857	2.29609	0.01695
0.38	44.87	0.68825	0.27994	2.41618	0.01829
0.34	48.25	0.72761	0.32082	2.55764	0.02000
0.3	52.62	0.77460	0.37669	2.72788	0.02236
0.25	60.61	0.84853	0.48659	3.00046	0.02715
0.23	65.36	0.88465	0.55577	3.13731	0.03036
0.21	72.20	0.92582	0.65884	3.29956	0.03559
0.195	81.53	0.96077	0.80327	3.45338	0.04423

Table B.2   (Continued)

β	φ	x/b	z/b	h/a	V/a³
0.1925	95.40	0.96694	1.01465	3.53793	0.06203
0.21	108.40	0.92582	1.19288	3.47260	0.08758
0.23	115.90	0.88465	1.27945	3.38272	0.10864
0.25	121.34	0.84853	1.33222	3.29944	0.12796
0.3	131.17	0.77460	1.40212	3.12503	0.17414
0.34	137.09	0.7276	1.42704	3.01374	0.21045
0.38	142.08	0.68825	1.43800	2.92095	0.24666
0.42	146.46	0.65465	1.44035	2.84223	0.28285
0.46	150.41	0.62554	1.43727	2.77443	0.31905
0.50	154.05	0.60000	1.43052	2.71526	0.35524
0.58	160.65	0.55709	1.41014	2.61634	0.42743
0.66	166.63	0.52223	1.38465	2.53620	0.49919
0.7	169.45	0.50709	1.37088	2.50133	0.53471
0.8	176.18	0.47434	1.33487	2.42538	0.62303
0.9	182.56	0.44721	1.29805	2.36147	0.70984
1.0	188.74	0.42426	1.26124	2.30604	0.79517
1.1	194.79	0.40452	1.22489	2.25680	0.87863
1.2	200.82	0.38730	1.18910	2.21206	0.96038
1.5	219.70	0.34641	1.08403	2.09350	1.19399
1.7	234.58	0.32540	1.01218	2.01784	1.33856
1.7	304.88	0.32540	0.83893	1.85810	1.29849
1.2	337.48	0.38730	0.87818	1.97123	0.91837
1.1	343.30	0.40452	0.89730	2.01386	0.84032
1.0	349.15	0.42426	0.92136	2.06571	0.76142
0.9	355.15	0.44721	0.95155	2.12903	0.68168

Table B.3. Size, Shape, Pressure, and Volume of Bubbles and
Drops with Given Radius of Attachment

$r/a = 0.4$

β	ϕ_r	x/b	z/b	h/a	V/a^3
30.0	6.17	0.10328	0.00548	0.27941	0.00533
18.0	7.98	0.13333	0.00912	0.36068	0.00692
12.0	9.79	0.16330	0.01372	0.44187	0.00850
8.0	12.02	0.20000	0.02063	0.54126	0.01047
5.5	14.54	0.24121	0.03019	0.65309	0.01270
4.0	17.13	0.28284	0.04173	0.76612	0.01504
3.5	18.35	0.30237	0.04787	0.81926	0.01613
2.9	20.24	0.33218	0.05808	0.90040	0.01786
2.3	22.86	0.37300	0.07385	1.01170	0.02030
2.0	24.63	0.40000	0.08546	1.08546	0.02196
1.8	26.06	0.42164	0.09548	1.14468	0.02332
1.5	28.78	0.46188	0.11589	1.25507	0.02594
1.25	31.84	0.50596	0.14108	1.37644	0.02897
1.1	34.23	0.53936	0.16228	1.46875	0.03139
1.0	36.17	0.56569	0.18038	1.54176	0.03341
0.8	41.32	0.63246	0.23257	1.72823	0.03897
0.66	46.69	0.69631	0.29255	1.90883	0.04512
0.58	50.97	0.74278	0.34404	2.04222	0.05041
0.5	56.89	0.80000	0.41997	2.20998	0.05828
0.46	60.93	0.83406	0.47434	2.31263	0.06413
0.42	66.32	0.87287	0.54940	2.43395	0.07262
0.38	74.69	0.91766	0.66956	2.58601	0.08785
0.36	97.25	0.94281	0.98565	2.77520	0.14837
0.38	105.99	0.91766	1.09093	2.76968	0.18416
0.42	115.44	0.87287	1.18463	2.72505	0.23502
0.46	121.96	0.83406	1.23412	2.67701	0.27950
0.5	127.16	0.80000	1.26371	2.63185	0.32143
0.58	135.45	0.74278	1.12924	2.55291	0.40169
0.66	142.15	0.69631	1.29953	2.48730	0.47925
0.7	145.12	0.67612	1.29854	2.45854	0.51723
0.8	151.82	0.63246	1.28804	2.39577	0.61073
0.9	157.75	0.59628	1.27074	2.34315	0.70194
1.0	163.17	0.56569	1.24977	2.29794	0.79124
1.1	168.21	0.53936	1.22692	2.25831	0.87839

Table B.3   (Continued)

β	φ	x/b	z/b	h/a	V/a³
1.2	172.98	0.51640	1.20312	2.22293	0.96372
1.25	175.29	0.50596	1.19105	2.20652	1.00587
1.5	186.20	0.46188	1.13075	2.13396	1.20846
1.7	194.51	0.43386	1.08367	2.08375	1.36230
2.0	206.81	0.40000	1.01557	2.01557	1.57998
2.5	229.01	0.35777	0.90542	1.90671	1.90695
2.5	310.00	0.35777	0.71442	1.69317	1.81377
2.0	331.06	0.40000	0.73401	1.73401	1.47971
1.7	342.67	0.43386	0.76316	1.78825	1.27326
1.5	350.54	0.46188	0.79225	1.84081	1.13175

Table B.4. Size, Shape, Pressure, and Volume of Bubbles and
Drops with Given Radius of Attachment

r/a = 0.6

β	ϕ_r	x/b	z/b	h/a	V/a^3
60.0	6.98	0.11094	0.00647	0.21801	0.01748
30.0	9.75	0.15492	0.01266	0.30724	0.02813
18.0	12.64	0.20000	0.02120	0.39692	0.03656
12.0	15.54	0.24495	0.03198	0.48658	0.04515
8.0	19.17	0.30000	0.04838	0.59676	0.05604
5.5	23.35	0.36181	0.07133	0.72131	0.06879
4.0	27.72	0.42426	0.09970	0.84811	0.08255
3.5	29.83	0.45356	0.11501	0.90807	0.08935
2.9	33.16	0.49827	0.14101	1.00025	0.10029
2.3	37.95	0.55950	0.18242	1.12813	0.11668
2.0	41.31	0.60000	0.21408	1.21407	0.12865
1.8	44.15	0.63246	0.24231	1.28397	0.13910
1.7	45.82	0.65079	0.25954	1.32394	0.14544
1.5	49.88	0.69282	0.30298	1.41709	0.16135
1.25	57.15	0.75895	0.38608	1.57014	0.19222
1.2	59.11	0.77460	0.40935	1.60808	0.20112
1.1	63.92	0.80904	0.46757	1.69516	0.22420
1.0	70.86	0.84853	0.55331	1.80546	0.26115
1.0	110.89	0.84853	0.97076	2.10065	0.61473
1.1	119.62	0.80904	1.01944	2.10444	0.74147
1.2	126.22	0.77460	1.04278	2.09873	0.85283
1.25	129.07	0.75895	1.04934	2.09449	0.90534
1.5	140.75	0.69282	1.05581	2.06906	1.14754
1.7	148.28	0.65079	1.04488	2.04798	1.32506
1.8	151.67	0.63246	1.03679	2.37679	1.40998
2.0	157.90	0.60000	1.01784	2.01784	1.57280
2.3	166.26	0.55950	0.98579	1.98964	1.80255
3.0	183.08	0.48990	0.90845	1.92912	2.28289
3.5	193.87	0.45356	0.85574	1.88796	2.58703
4.0	204.25	0.42426	0.80597	1.84692	2.86294
6.0	258.64	0.34641	0.60326	1.62222	3.68271
6.0	281.23	0.34641	0.55639	1.54104	3.59171
4.0	332.13	0.42426	0.55032	1.48537	2.56096
3.0	351.28	0.48990	0.59863	1.54966	2.03835

Table B.5. Size, Shape, Pressure, and Volume of Bubbles and
Drops with Given Radius of Attachment

$r/a = 1.0$

β	ϕ_r	x/b	z/b	h/a	V/a^3
60.0	13.46	0.18257	0.01914	0.28739	0.17182
30.0	19.24	0.25820	0.03877	0.40835	0.24755
18.0	25.24	0.33333	0.06585	0.53089	0.32807
16.0	26.91	0.35355	0.07454	0.56438	0.35097
12.0	31.60	0.40825	0.10137	0.65655	0.41635
8.0	40.18	0.50000	0.15888	0.81776	0.54200
6.5	45.96	0.55470	0.20262	0.91998	0.63182
5.5	51.73	0.60302	0.24929	1.01642	0.72663
4.5	61.02	0.66667	0.32813	1.15886	0.89237
4.0	69.09	0.70711	0.39734	1.26903	1.05200
3.7	77.81	0.73522	0.46919	1.37338	1.24390
3.7	103.15	0.73522	0.63176	1.59450	1.95007
4.0	113.93	0.70711	0.67052	1.65536	2.32888
4.5	125.33	0.66667	0.69002	1.70170	2.78302
5.5	140.90	0.60302	0.68546	1.73973	3.48414
6.5	152.52	0.55470	0.66411	1.75195	4.05442
8.0	166.51	0.50000	0.62550	1.75100	4.76824
10.0	181.82	0.44721	0.57493	1.73279	5.54361
16.0	220.40	0.35355	0.44901	1.62354	7.13667
20.0	253.01	0.31623	0.36935	1.48421	7.66733
20.0	286.54	0.31623	0.32147	1.33281	7.19870
16.0	316.08	0.35355	0.31169	1.23514	6.05930
10.0	348.47	0.44721	0.35006	1.22997	4.49208

Table B.6. Size, Shape, Pressure, and Volume of Bubbles and
Drops with Given Radius of Attachment

$r/a = 1.5$

β	ϕ_r	x/b	z/b	h/a	V/a^3
100.0	21.00	0.21213	0.03058	0.35762	0.83884
80.0	23.69	0.23717	0.03859	0.40220	0.94928
60.0	27.71	0.27386	0.05210	0.46793	1.11632
40.0	34.97	0.33541	0.08054	0.58381	1.42593
30.0	41.80	0.38730	0.11119	0.68884	1.72815
24.0	48.69	0.43301	0.14478	0.79022	2.04596
20.0	56.13	0.47434	0.18271	0.89402	2.40649
18.0	61.81	0.50000	0.21185	0.96887	2.69503
16.0	70.85	0.53033	0.25677	1.07982	3.18127
18.0	124.73	0.50000	0.39672	1.52349	6.89629
20.0	133.98	0.47434	0.39444	1.56354	7.66084
24.0	148.02	0.43301	0.38033	1.60617	8.85724
30.0	163.84	0.38730	0.35409	1.62959	10.20717
50.0	200.61	0.30000	0.27914	1.59568	12.93770
64.0	222.48	0.26517	0.23868	1.52697	13.97587
80.0	255.04	0.23717	0.19292	1.37824	14.29483
80.0	284.50	0.23717	0.16827	1.22238	13.20281
64.0	313.14	0.26517	0.16132	1.08935	11.13856
50.0	330.62	0.30000	0.16778	1.03892	9.65580
32.0	354.65	0.37500	0.19654	1.03617	7.76366

C. COMPUTER PROGRAM

The Fortran listing of the computer program for solving the interfacial equation and computing the various parameters presented in this report is given in Table C.1. The ranges of ϕ, β, and r/a can be easily extended at some sacrifice of either the running time or the accuracy. Other parameters, such as maximum drop height and radius as a function of drop volume and contact angle, can be obtained either by modifying one of the MAIN subroutines or by adding another routine.

Table C.1. Computer Program

```
C SOLUTION OF INTERFACIAL EQ. FOR THE PRESS. AND VOL.
C WITHIN ATTACHED BUBBLE. CASE 1-ATTACHED ABOVE (POSITIVE BETA)
C
      IMPLICIT REAL*8(D),REAL*4(A-C,E-H,O-Z)
      DIMENSIONA(4),B(4),C(4),D(4),E(8),R(4),G1(10),P(4)
      DIMENSIONDA(4),DB(4),DC(4),DZ(4),DE(8),DP(4),DR(4)
    9 FORMAT (I1,I2)
   11 READ  9,I,N3
      IF (I-1)401,10,501
C
C SOLUTION AT MAX.PRESS FOR GIVEN VALUES OF R/A(=G)
C
  300 FORMAT(F15.7)
   10 CONTINUE
      DO 201 N4=1,N3
      READ300,G
C ESTIMATE OF INITIAL BETA
      IF(G.GE.0.099.AND.G.LT.0.4)GOTO1001
      IF(G.GE.0.39.AND.G.LT.0.8)GOTO1002
      IF(G.GE.0.79.AND.G.LT.3.01)GOTO1003
 1001 BETA=.11175-.92434*G+3.92521*G**2.
      GOTO301
 1002 BETA=2.87267-12.78143*G+16.40021*G**2.
      GOTO301
 1003 BETA=16.30443-49.43188*G+40.54628*G**2.
      GOTO301
C COURSE BETA GRID
  301 BETA=BETA+0.1*BETA
      N20=0
      P(2)=0
   58 BETA=BETA-.01*BETA
      CALL MAINS(A,B,C,Z,E,H,J,P,BETA,G,R)
      IF(P(1)-P(2))59,59,60
   60 P(2)=P(1)
      N20=N20+1
      GOTO58
   59 IF(N20-1)301,301,57
   57 PRINT 50,BETA,H
      PRINT 100
      PRINT 200,E(8),E(6),E(7),R(1),P(1),V
      BETA=BETA+.02*BETA
      P(2)=0
C FINE BETA GRID
   65 BETA=BETA-.001*BETA
      CALL MAINS(A,B,C,Z,E,H,J,P,BETA,G,R)
      IF(P(1)-P(2))61,61,62
   62 P(2)=P(1)
      GOTO65
```

Table C.1 (Continued)

```
   61 PRINT 50,BETA,H
      PRINT 100
      PRINT 200,E(8),E(6),E(7),R(1),P(1),V
      BETA=BETA+.002*BETA
      DP(2)=0
C EXTRA FINE BETA GRID WITH DOUBLE PRECISION
      DBETA=BETA
      DG=G
   66 DBETA=DBETA-.0001*DBETA
      CALL MAIND(DA,DB,DC,DZ,DE,DH,J,DP,DBETA,DG,DR)
      IF(DP(1)-DP(2))63,63,64
   64 DP(2)=DP(1)
      GOTO66
   63 DV=3.14159265*DG   *(DP(1)*DG   -DSIN(DE(5)))
      DE(4)=DE(1)*57.29577957
      DE(8)=DE(5)*57.29577957
      PRINT 50,DBETA,DH
   50 FORMAT(1H2,6HBETA =,E15.7,6X,3HH =,E15.7///)
  100 FORMAT(1H0,5X,3HPHI,8X,3HX/B,13X,3HZ/B,13X,3HR/A,13X,3HH/A,
     113X,4HV/A3)
  200 FORMAT(1H ,0P1F9.3,1P5E16.7)
      PRINT100
      PRINT200,DE(8),DE(6),DE(7),DG,DP(1),DV
  201 CONTINUE
      GOTO 11
C
C SOLUTION FOR GIVEN VALUES OF BETA AND R/A(=G1)
C
  400 FORMAT(8F10.7)
  401 CONTINUE
      DO 202 N4=1,N3
      READ 400,BETA,(G1(I),I=1,8)
      CALL MAINIS(A,B,C,Z,E,H,J,P,BETA,G1,R)
  202 CONTINUE
      GOTO 11
C
C SOLUTION FOR GIVEN VALUES OF BETA AT INTERVALS (N1) OF PHI
C                                                    UP TO N2
  501 CONTINUE
      DO 203 N4=1,N3
      READ 500,BETA,N1,N2
  500 FORMAT(F10.7,2I4)
      CALL MAIN2S(A,B,C,Z,E,H,J,P,BETA,G,R,N1,N2)
  203 CONTINUE
      GOTO 11
      END
C
C
C
```

Table C.1   (Continued)

```
      SUBROUTINE MAINS(A,B,C,Z,E,H,J,P,BETA,G,R)
C
      IMPLICIT REAL*8(D),REAL*4(A-C,E-H,O-Z)
      DIMENSIONA(4),B(4),C(4),Z(4),E(8),P(4),R(4)
      H=0.1745329E-01
      E(6)=0.
      J=3
      E(1)=0.
      E(2)=0.
      E(3)=0.
      N1=1
      R(2)=0.
      DO150M=1,360
      CALL RUNGKS(A,B,C,Z,E,H,J,BETA,G,P)
      R(1)=E(2)*SQRT(BETA/2.0)
      GOTO(4,5),N1
    4 IF(G-R(1))5,3,3
    3 IF(R(1)-R(2))8,8,10
    5 IF(G-R(1))1,6,7
    1 N1=2
   10 R(2)=R(1)
      E(5)=E(1)
      E(6)=E(2)
      E(7)=E(3)
  150 CONTINUE
    7 F=(G-R(2))/(R(1)-R(2))
      E(5)=F*(E(1)-E(5))+E(5)
      E(6)=F*(E(2)-E(6))+E(6)
      E(7)=F*(E(3)-E(7))+E(7)
    6 P(1)=SQRT(2./BETA)+E(7)*SQRT(BETA/2.)
      GOTO9
    8 P(1)=1.0
      P(2)=2.0
    9 RETURN
      END
C
C
C
      SUBROUTINE MAIND(DA,DB,DC,DZ,DE,DH,J,DP,DBETA,DG,DR)
C
      IMPLICIT REAL*8(D),REAL*4(A-C,E-H,O-Z)
      DIMENSIONDA(4),DB(4),DC(4),DZ(4),DE(8),DP(4),DR(4)
      DH=0.2726646D-02
      DE(6)=0.
      J=3
      DE(1)=0.
      DE(2)=0.
      DE(3)=0.
      N1=1
```

Table C.1 (Continued)

```
      DO150M=1,720
      CALL RUNGKD(DA,DB,DC,DZ,DE,DH,J,DBETA,DG,DP)
      DR(1)=DE(2)*DSQRT(DBETA/2.0)
      GOTO(4,5),N1
    4 IF(DG-DR(1))5,150,150
    5 IF(DG-DR(1))1,6,7
    1 DR(2)=DR(1)
      DE(5)=DE(1)
      DE(6)=DE(2)
      DE(7)=DE(3)
      N1=2
  150 CONTINUE
    7 DF=(DG-DR(2))/(DR(1)-DR(2))
      DE(5)=DF*(DE(1)-DE(5))+DE(5)
      DE(6)=DF*(DE(2)-DE(6))+DE(6)
      DE(7)=DF*(DE(3)-DE(7))+DE(7)
    6 DP(1)=DSQRT(2./DBETA)+DE(7)*DSQRT(DBETA/2.)
      RETURN
      END
C
C
C


      SUBROUTINE MAINIS(A,B,C,Z,E,H,J,P,BETA,G1,R)
C
      IMPLICIT REAL*8(D),REAL*4(A-C,E-H,O-Z)
      DIMENSIONA(4),B(4),C(4),Z(4),E(8),P(4),R(4),G1(10)
      H=0.8726646E-02
      E(6)=0.
      J=3
      R(2)=0.
      E(1)=0.
      E(2)=0.
      E(3)=0.
      N2=1
      N1 = 1
      I=1
      DO 150 M=1,720
      CALL RUNGKS(A,B,C,Z,E,H,J,BETA,G1,P)
      R(1)=E(2)*SQRT(BETA/2.0)
      GOTO(4,5,22),N1
    4 IF(G1(I)-R(1))7,6,3
    3 IF(R(1)-R(2))10,10,140
   10 I=I-1
    5 N2=-1
      N1=2
      IF(G1(I)-R(1))20 ,6,7
   20 IF(R(1)-R(2))140,21,21
   21 I=I+1
```

Table C.1 (Continued)

```
 22 N2=1
    N1=3
    IF(G1(I)-R(1))7,6,140
  7 F=(G1(I)-R(2))/(R(1)-R(2))
    E(1)=F*(E(1)-E(5))+E(5)
    E(2)=F*(E(2)-E(6))+E(6)
    E(3)=F*(E(3)-E(7))+E(7)
  6 P(1)=SQRT(2./BETA)+E(3)*SQRT(BETA/2.)
    V=3.14159265*G1(I)*(P(1)*G1(I)-SIN(E(1)))
     E(4)= E(1)*57.29577957
 50 FORMAT(1H2,6HBETA =,E15.7,6X,3HH =,E15.7///)
    PRINT 50,BETA,H
    PRINT 100
    PRINT 200,E(4),E(2),E(3),G1(I),P(1),V
200 FORMAT(1H  ,0P1F9.3,1P5E16.7)
100 FORMAT(1H0,5X,3HPHI,8X,3HX/B,13X,3HZ/B,13X,3HR/A,13X,3HH/A,
   113X,4HV/A3)
    I=I+N2
    IF(I-8)130,130,3
130 IF(I-1)9,140,140
140 R(2)=R(1)
    E(5)=E(1)
    E(6)=E(2)
    E(7)=E(3)
150 CONTINUE
  9 RETURN
    END
C
C
C


    SUBROUTINE MAIN2S(A,B,C,Z,E,H,J,P,BETA,G,R,N1,N2)
C
    IMPLICIT REAL*8(D),REAL*4(A-C,E-H,O-Z)
    DIMENSIONA(4),B(4),C(4),Z(4),E(8),P(4),R(4)
101 FORMAT(1H0,5X,3HPHI,8X,3HX/B,13X,3HZ/B,13X,3HR/A,13X,3HH/A,
   113X,4HV/A3)
 50 FORMAT(1H1,5HBETA=,E15.7,6X,2HH=,E15.7///)
200 FORMAT(1H  ,0P1F9.3,1P5E16.7)
    H=0.1745329E-01
    PRINT 50,BETA,H
    PRINT 101
    E(6)=0.
    J=3
    N2=N2/N1
    N3=.1746E-01/H
    N1=N3*N1
    E(1)=0.
    E(2)=0.
    E(3)=0.
    DO 150 M=1,N2
```

Table C.1 (Continued)

```
      DO 100 N=1,N1
      CALL RUNGKS(A,B,C,Z,E,H,J,BETA,G,P)
  100 CONTINUE
      G=E(2)*SQRT(BETA/2.0)
      E(6)=E(2)
      E(7)=E(3)
      P(1)=SQRT(2./BETA)+E(7)*SQRT(BETA/2.)
      V=3.141593*G*(P(1)*G-SIN(E(1)))
      E(4)=E(1)*57.29577957
      PRINT 200,E(4),E(6),E(7),G,P(1),V
  150 CONTINUE
      RETURN
      END
C
C
C

      SUBROUTINERUNGKD(DA,DB,DC,DZ,DE,DH,J,DBETA,DG,DP)
C
      IMPLICIT REAL*8(D),REAL*4(A-C,E-H,O-Z)
      DIMENSIONDA(4),DB(4),DC(4),DZ(4),DE(8),DP(4),DR(4)
      DA(1)=DH*0.16666666667
      DA(2)=DH*0.33333333333
      DA(3)=DA(2)
      DA(4)=DA(1)
      DB(4)=0.
      DB(1)=DH*0.5
      DB(2)=DB(1)
      DB(3)=DH
      DO20I=1,J
      DC(I)=0.
   20 DZ(I)=DE(I)
      DO30K=1,4
      CALL RHOD(DA,DB,DC,DZ,DE,DH,J,DBETA,DG,DP)
      DZ(1)=1.
      DO50I=1,J
      DC(I)=DC(I)+DA(K)*DZ(I)
   50 DZ(I)=DE(I)+DB(K)*DZ(I)
   30 CONTINUE
      DC(1)=DH
      DO60I=1,J
   60 DE(I)=DE(I)+DC(I)
      RETURN
      END
C
C
C

      SUBROUTINE RHOD (DA,DB,DC,DZ,DE,DH,J,DBETA,DG,DP)
C
      IMPLICIT REAL*8 (D),REAL*4(A-C,E-H,O-Z)
      DIMENSIONDA(4),DB(4),DC(4),DZ(4),DE(8),DP(4),DR(4)
```

Table C.1   (Continued)

```
      IF(DZ(1))70,75,70
70    DRH=1.0/(2.0+DBETA*DZ(3)-DSIN(DZ(1))/DZ(2))
      GOTO76
75    DRH=1.0
76    DZ(2)=DRH*DCOS(DZ(1))
      DZ(3)=DRH*DSIN(DZ(1))
      RETURN
      END
C
C
C


      SUBROUTINERUNGKS(A,B,C,Z,E,H,J,BETA,G,P)
C
      IMPLICIT REAL*8(D),REAL*4(A-C,E-H,O-Z)
      DIMENSIONA(4),B(4),C(4),Z(4),E(8),P(4),R(4)
      A(1)=H*0.16666666667
      A(2)=H*0.33333333333
      A(3)=A(2)
      A(4)=A(1)
      B(4)=0.
      B(1)=H*0.5
      B(2)=B(1)
      B(3)=H
      DO20I=1,J
      C(I)=0.
20    Z(I)=E(I)
      DO30K=1,4
      CALLRHOS(A,B,C,Z,E,H,J,BETA,G,P)
      Z(1)=1.
      DO50I=1,J
      C(I)=C(I)+A(K)*Z(I)
50    Z(I)=E(I)+B(K)*Z(I)
30    CONTINUE
      C(1)=H
      DO60I=1,J
60    E(I)=E(I)+C(I)
      RETURN
      END
C
C
C


      SUBROUTINERHOS(A,B,C,Z,E,H,J,BETA,G,P)
C
      IMPLICIT REAL*8(D),REAL*4(A-C,E-H,O-Z)
      DIMENSIONA(4),B(4),C(4),Z(4),E(8),P(4),R(4)
      IF(Z(1))70,75,70
70    RH=1.0/(2.0+BETA*Z(3)-SIN(Z(1))/Z(2))
      GOTO76
75    RH=1.0
76    Z(2)=RH*COS(Z(1))
      Z(3)=RH*SIN(Z(1))
      RETURN
      END
```

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